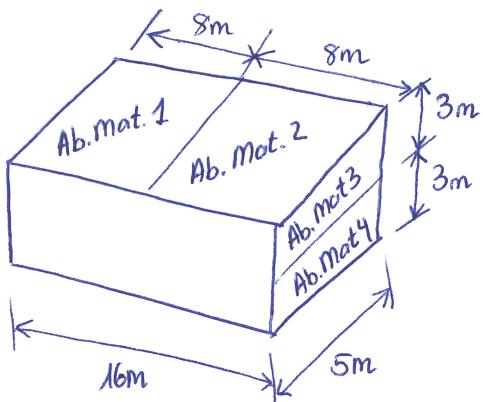


EXAM VTAN01 - 12th JANUARY 2017

(1)



Material	α
Ab. Mat 1	0'24
Ab. Mat 2	0'37
Ab. Mat 3	0'15
Ab. Mat 4	0'22

NOTE: α is given for diffuse field conditions
(i.e. average for all freqs)

- (a) Reverberation time is the time required for the reflections of a direct sound to decay 60 dB. Two procedures to measure it were employed in Lab 2.

- 1) Interrupted noise: loudspeaker playing pink noise / all freq. excited with same energy
- 2) Impulse excitation: book banged or clap / some freqs. may not be excited

(b) Sabine's formula: $T_{60} = 0'16 \cdot \frac{V}{A_{eff}}$; with $A_{eff} = \sum_i \alpha_i S_i$

We can first calculate the effective absorption area separately

$$A_{eff\text{longwalls}} = A_{eff\text{floor}} = A_{eff\text{frontwall}} = 0 \quad (\text{totally reflective; i.e. } \alpha = 0)$$

$$A_{eff\text{ceiling}} = \frac{\text{Ceiling}}{2} \cdot \alpha_{\text{Ab. mat 1}} + \frac{\text{Ceiling}}{2} \cdot \alpha_{\text{Ab. mat 2}} = (8 \cdot 5 \cdot 0'24) + (8 \cdot 5 \cdot 0'37) = 24'4 \text{ m}^2 \text{ Sab.}$$

$$A_{eff\text{wall}} = \frac{\text{Wall}}{2} \cdot \alpha_{\text{Ab. mat 3}} + \frac{\text{Wall}}{2} \cdot \alpha_{\text{Ab. mat 4}} = (3 \cdot 5 \cdot 0'15) + (3 \cdot 5 \cdot 0'22) = 5'55 \text{ m}^2 \text{ Sab.}$$

- Thus, the total effective absorption area is: $A_{eff} = \sum_i \alpha_i S_i = 24'4 + 5'55 = 29'95 \text{ m}^2 \text{ Sab.}$

- The volume of the room is: $V_{room} = 16 \cdot 5 \cdot 6 = 480 \text{ m}^3$

Hence, the reverberation time is: $T_{60} = 0'16 \cdot \frac{480}{29'95} \Rightarrow T_{60} = 2'56 \text{ sec}$

The value would be high or low depending on the room's use.

(c)

$T'_{60} = 0'95 \cdot T_{60} = 2'43 \text{ sec.}$ is to be obtained now to the stipulated reduction

- The new effective area will now be:

$$A'_{eff} = A_{eff\text{ceiling}} + A_{eff\text{wall}} + A_{eff\text{new walls}} = 24'4 + 5'55 \text{ m}^2$$

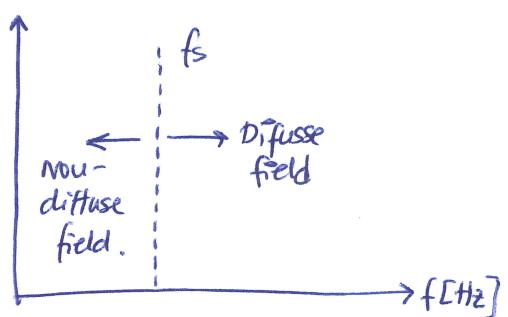
$$S_{new} \approx 5 \text{ m}^2$$

thus: $T'_{60} = 0'16 \cdot \frac{V_{room}}{A'_{eff}} \Rightarrow A'_{eff} = 0'16 \cdot \frac{V_{room}}{T'_{60}} = 24'45 + 0'35 \cdot S_{new} = 0'16 \cdot \frac{480}{2'43} \Rightarrow$

(d) $f_s = 2000 \sqrt{\frac{T_{60}}{V_{room}}} = 2000 \cdot \sqrt{\frac{2'43}{480}} = 142'3 \text{ Hz}$

Diffuse field: SPL \approx constant in the room

Nou-diffuse field: wave approach



- (2) a) 10 km far away (point 1): $L_{I_1} = 10 \log \left(\frac{I_1}{I_0} \right) = 70 \text{ dB}$
 1 km distance (point 2): $L_{I_2} = 10 \log \left(\frac{I_2}{I_0} \right)$

The intensity of the sound wave at the deck is:

$$I_0 = 10 \log \left(\frac{I_1}{10^{12}} \right) \Rightarrow I_1 = 10^{12} \cdot 10^7 = 10^{-5} \frac{\text{W}}{\text{m}^2}$$

Due to spherical propagation being assumed: $\frac{I_2}{I_1} = \frac{r_1^2}{r_2^2} \Rightarrow I_2 = I_1 \left(\frac{r_1^2}{r_2^2} \right) \Rightarrow$
 $\Rightarrow I_2 = 10^{-5} \cdot \left(\frac{10^2}{(10^3)^2} \right) \Rightarrow I_2 = 10^{-9} \frac{\text{W}}{\text{m}^2}$

Thus, the sound intensity is: $L_{I_2} = 10 \log \left(\frac{I_2}{I_0} \right) \Rightarrow L_{I_2} = 30 \text{ dB}$

- (b) The distance at which the sound stops being audible is that at which the intensity equals the threshold value, $I_0 = 10^{-12} \frac{\text{W}}{\text{m}^2}$.

Due to spherical propagation: $\frac{I_1}{I_0} = \frac{r_0^2}{r_1^2} \Rightarrow r_0 = r_1 \sqrt{\frac{I_1}{I_0}} = 10 \sqrt{\frac{10^{-5}}{10^{-12}}} \Rightarrow r_0 = 31600 \text{ m}$

(c) $I = \frac{\tilde{P}^2}{P_c} \Rightarrow \tilde{P} = \sqrt{P_c \cdot I_0} = \sqrt{12 \cdot 340 \cdot 10^{-12}} \Rightarrow \tilde{P} = 2 \cdot 10^{-5} \text{ Pa}$

It does make sense; pressure threshold ($20 \mu\text{Pa}$) \approx intensity threshold.

(d) $L_p = 20 \log \left(\frac{(20 \mu\text{Pa})^2}{(20 \mu\text{Pa})^2} \right) = 0 \text{ dB}$

- (3) a) Pure tone \Rightarrow harmonic wave; $p(x,t) = P \cos(kx - \omega t + \varphi)$

We calculate the terms separately:

$$\omega = 2\pi \cdot f = 2\pi \cdot 612 = 1224\pi \text{ Hz}$$

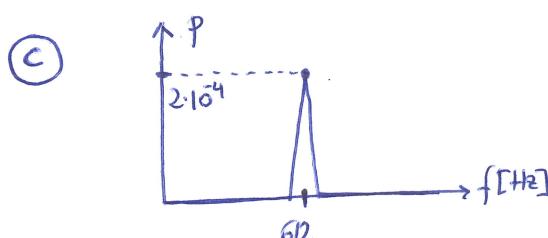
$$k = \frac{\omega}{c} = \frac{1224\pi}{340} = 36\pi \text{ m}^{-1}$$

$$\lambda = \frac{2\pi}{k} = \frac{2\pi}{36\pi} = 0.555 \text{ m}$$

Considering $\varphi = 0$

$$p(x,t) = (2 \cdot 10^{-4}) \cos(36\pi x - 1224\pi t)$$

(b) $L_p = 20 \log \left(\frac{\tilde{P}^2}{P_{ref}^2} \right) = 20 \log \left(\frac{\left(\frac{2 \cdot 10^{-4}}{\sqrt{2}} \right)^2}{(2 \cdot 10^{-5})^2} \right) \Rightarrow L_p = 40 \text{ dB}$



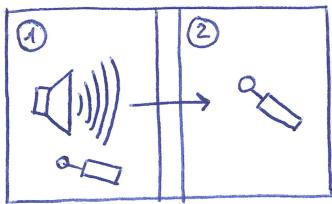
The leap between the time and the frequency domain would be performed by means of Fourier Transform analyses.

- (d) Sampling frequency should be double as the highest freq. of interest
 Thus: $f_s > 2 \cdot 612 = 1224 \text{ Hz} \rightarrow$ Nyquist-Shannon criterion.

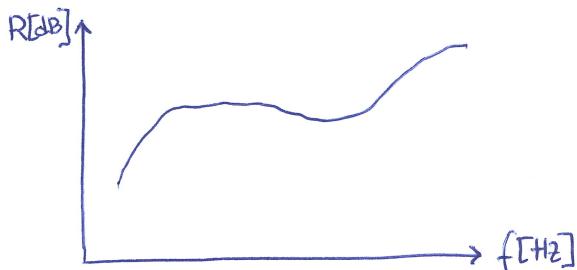
(e) $L_{Pr} = 10 \log \left(\sum_i 10 \frac{L_{Pi}}{10} \right) = 10 \log \left(4 \cdot 10^{\frac{40}{10}} \right) \Rightarrow L_{Pr} = 46 \text{ dB}$

(4)

a)



$$R(f) = L_{\text{send}}(f) - L(f)_{\text{rec}} + 10 \log \left(\frac{S}{A_{\text{eff}}} \right)$$



- Play sound (pink noise: equal energy in all frequencies) in sending room (room 1)

- measure SPL in both room ① and ②
- Different number of microphone positions and loudspeaker positions should be considered → given in the standards).

- measure the reverberation time in the receiving room in order to obtain the effective absorption area of the room. Two methods:

- interrupted noise
- impulse excitation

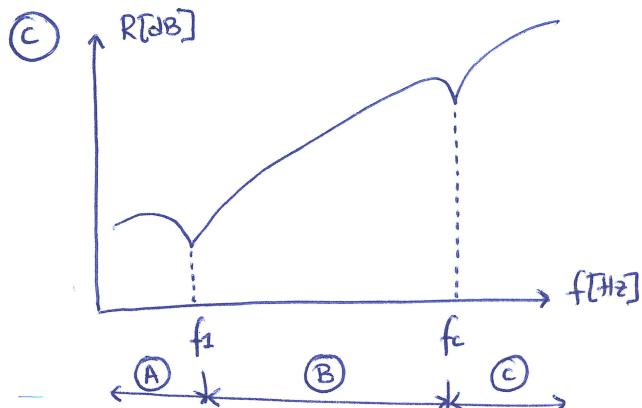
Explained in 1a

b) Due to the fact that $V_2 = 50 \text{ m}^2$ and that $T_{60,2} = 0.8 \text{ s}$. for all frequencies we have that:

$$A_{\text{eff}} = 0.16 \cdot \frac{V_2}{T_{60,2}} = 0.16 \cdot \frac{50}{0.8} = 10 \text{ m}^2 \text{ Sab.}$$

And since the surface of the wall is 10 m^2 , the term $10 \log \left(\frac{S}{A_{\text{eff}}} \right)$ vanishes. Thus

$$R(f) = L_{\text{send}}(f) - L_{\text{rec}}(f)$$



$f [\text{Hz}]$	$R(f)$
50	25
63	27
80	28
100	30
125	31
160	33
200	35
250	37
:	:
2500	53
3150	55
4000	57
5000	59

→ PLOT!

f_1 = resonance frequency

f_c = coincidence frequency

- (A) Stiffness-controlled region
- (B) Mass-controlled region
- (C) Damping-controlled region.

d) No, we cannot say anything, since impact sound insulation measurements are performed using a tapping machine as excitation source, and not a loudspeaker.

e)

	$R'_{W,\text{wall}} = 44 \text{ dB}$
	$R'_{W,\text{wind}} = 25 \text{ dB}$
	$S_{\text{window}} = 2 \text{ m}^2$
	$S_{\text{wall}} = 10 \text{ m}^2$

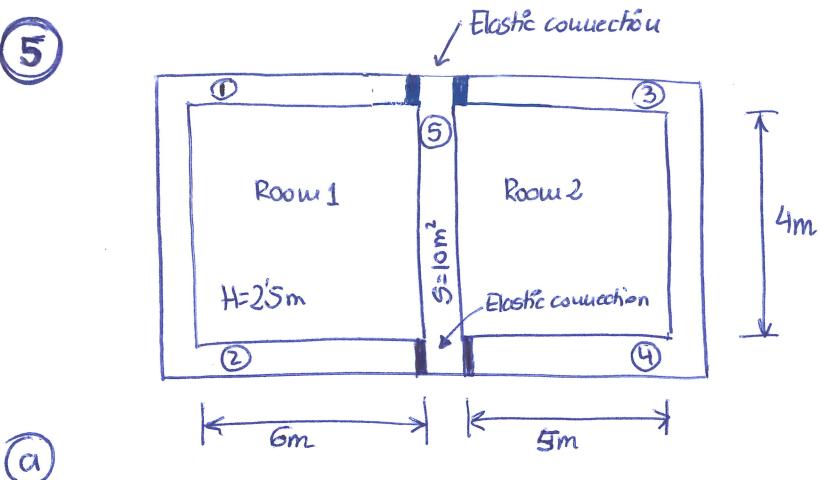
$$R'_{W,\text{Total}} = 10 \log \left[\sum_{i=1}^n S_i 10^{-\frac{R'_i}{10}} \right]; \text{ Then:}$$

$$R'_{W,\text{Total}} = 10 \log \left[\frac{1}{10} \left(8 \cdot 10^{\frac{44}{10}} + 2 \cdot 10^{\frac{25}{10}} \right) \right] \Rightarrow R'_{W,\text{Total}} = 43 \text{ dB}$$

43-4 = 39 dB

$$(f) R'_{W,\text{leak}} = -10 \log \left(10^{-\frac{R'_W}{10}} + \frac{S_{\text{leak}}}{S_{\text{Total}}} \right) \Rightarrow S_{\text{leak}} = S_{\text{Total}} \left(10^{-\frac{R'_W}{10}} - 10^{-\frac{R'_{W,\text{Total}}}{10}} \right) \Rightarrow S_{\text{leak}} = 7.5 \cdot 10^{-4} \text{ m}^2$$

5



$$R_{W,1} = R_{W,2} = R_{W,3} = R_{W,4} = 48 \text{ dB}$$

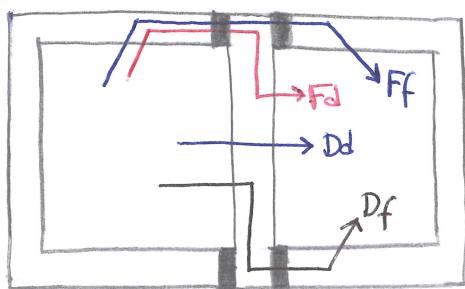
$$R_{W,S} = 52 \text{ dB}$$

For task (b) :

$$\alpha_{\text{Room}_1, \text{ceiling}} = 0.4$$

$$\alpha_{\text{Room}_2, \text{ceiling}} = 0.7$$

$$L_{\text{sending}} = 90 \text{ dB}$$



$$R_w' = -10 \log \left[10^{\frac{-R_{Df,w}}{10}} + \sum_{F=1}^n 10^{\frac{-R_{Ff,w}}{10}} + \sum_{F=2}^n 10^{\frac{-R_{Fd,w}}{10}} + \sum_{F=3}^n 10^{\frac{-R_{Dd,w}}{10}} \right]$$

with the R_{ij} paths defined as:

$$(R_f)_{ij,w} = R_{ij,w} = \frac{R_{i,w} + R_{j,w}}{2} + K_{ij} + 10 \log \left(\frac{S_s}{l_0 \cdot l_{ij}} \right)$$

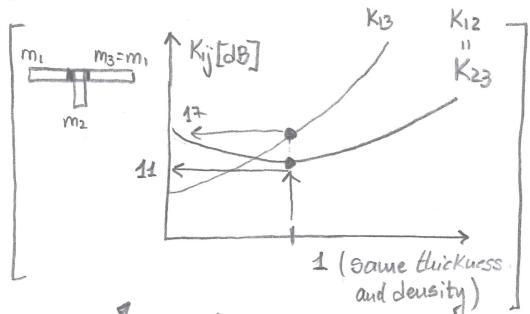
In our case (neglecting floor and ceiling flanking transmission), we have:

2 Df paths
2 Fd paths
2 Ff paths
1 Dd path

\Rightarrow In total, there are 7 reduction indexes to take into account.

(*) We first calculate the Df paths, black in the figure

$$R_{Df,w} = \frac{52+48}{2} + 11 + 10 \log \left(\frac{10}{1.25} \right) = 67 \text{ dB}$$



(*) The Ff paths (red colour in the figure) are calculated as:

$$R_{Ff,w} = \frac{48+52}{2} + 11 + 10 \log \left(\frac{10}{1.25} \right) = 67 \text{ dB}$$

(*) The Ff paths are:

$$R_{Ff,w} = \frac{48+48}{2} + 17 + 10 \log \left(\frac{10}{1.25} \right) = 71 \text{ dB}$$

$$\text{Thus, we have: } R_w' = -10 \log \left[10^{\frac{-52}{10}} + 4 \cdot 10^{\frac{-67}{10}} + 2 \cdot 10^{\frac{-71}{10}} \right] \Rightarrow R_w' = 51.4 \text{ dB}$$

(b) The sound pressure level in the receiving room is calculated as:

$$R_w' = L_{\text{sending}} - L_{\text{receiving}} + 10 \log \left(\frac{S}{A_2} \right) \Rightarrow L_{\text{receiving}} = L_{\text{sending}} - R_w' + 10 \log \left(\frac{S}{A_2} \right) \Rightarrow$$

$$\Rightarrow L_{\text{receiving}} = 90 - 51.4 + 10 \log \left(\frac{10}{(5 \cdot 4) \cdot 0.7} \right) \Rightarrow L_{\text{receiving}} = 37.1 \text{ dB}$$

⑥ a) Theoretical (see lecture notes)

b) Theoretical (see lecture notes)

c) Theoretical (see lecture notes)

d) • 60dBA for 12h and completely silent the rest

$$L_{\text{eq}, 24h} = 10 \log \left(\frac{1}{T} \int_0^T 10^{\frac{L_p(t)}{10}} dt \right) = 10 \log \left(\frac{1}{24} \int_0^{24} 10^{\frac{60}{10}} dt + 0 \right) = 57 \text{dBA}$$

$$\bullet L_{\text{eq}, 24h} = 10 \log \left(\frac{1}{24} \int_0^{0.25} 10^{\frac{95}{10}} dt + \int_0^{0.5} 10^{\frac{101}{10}} dt + 0 \right) = 84.70 \text{dB}$$

e) • $L_p = 20 \log \left(\frac{\tilde{P}}{P_{\text{ref}}} \right) \Rightarrow 0 = 20 \log \left(\frac{\tilde{P}}{P_{\text{ref}}} \right) \Rightarrow \tilde{P} = P_{\text{ref}} = 20 \mu\text{Pa}$

$$\bullet L_p = 20 \log \left(\frac{\tilde{P}}{P_{\text{ref}}} \right) \Rightarrow L_p = 20 \log \left(\frac{0}{P_{\text{ref}}} \right) \Rightarrow L_p = -\infty$$

