

recording

Rumsakustik

Erling Nilsson, Akustiker
ECOPHON Saint-Gobain





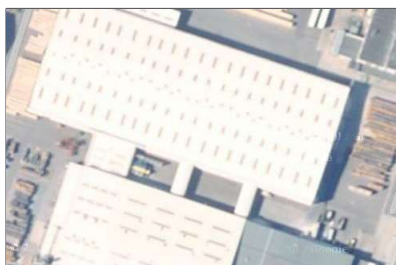
Community school no 15, Gdynia, Poland. Architect: Adam Drochomiercki. Photo: Szymon Polanski.
System: Ecophon Master A/alpha

Ecophon®

European supply chain



Næstved



Chalon



Hyllinge



Forssa



Gliwice

-  Production unit
-  Distribution center

Saint-Gobain

- One of the world's 100 leading industry groups
- Focusing on habitat and construction
- Established in 1665
- Present in 64 countries
- 190 000 employees
- ~ €40 billion in sales



Spegelsalen i Versailles

Four market segments



Education



Modern Office



Healthcare



Clean Industry

- Long experience of how sound affects people
- Specialised knowledge about segment specific activities
- Systems developed for specific needs

Benefits of good acoustics

- Increased wellbeing and satisfaction
- Less tiredness
- Easier to concentrate
- Fewer errors
- Less stress hormones
- Easier to communicate
- More positive energy
- Increased creativity



Innehåll

- Något om Ecophon
- Rumsakustik i praktiken
- Betydelsen av god akustik
- Rumsakustik och ljudabsorption
- “Activity based acoustic design”
- Rumsakustiska mått
- Effekt av akustikreglering i klassrum
- Öppna kontorslandskap
- Beräkning av rumsakustiska mått

Fö 2020-05-13, 13-15
Öv 2020-05-14, 10-12

Fö 2020-05-18, 10-12
Öv 2020-05-18, 13-15

Room Acoustic design in practise



Schools:



Positive effects of a good sound environment in educational premises include:

- Reduced vocal strain and voice disorders for teachers
- Improved concentration
- Reduced tiredness, fatigue and stress levels
- Easier to hear and be heard with improved speech clarity
- Optimised environment for multi-communicational activities such as group work
- Improved student behaviour and reduced burden on school and classroom management

Healthcare:

Better sound environment contributes to:

- Lowering of blood pressure
- Improving quality of sleep
- Reducing intake of pain medication
- Reducing the number of re-admissions
- Improving the wellbeing of staff and increasing perceived performance

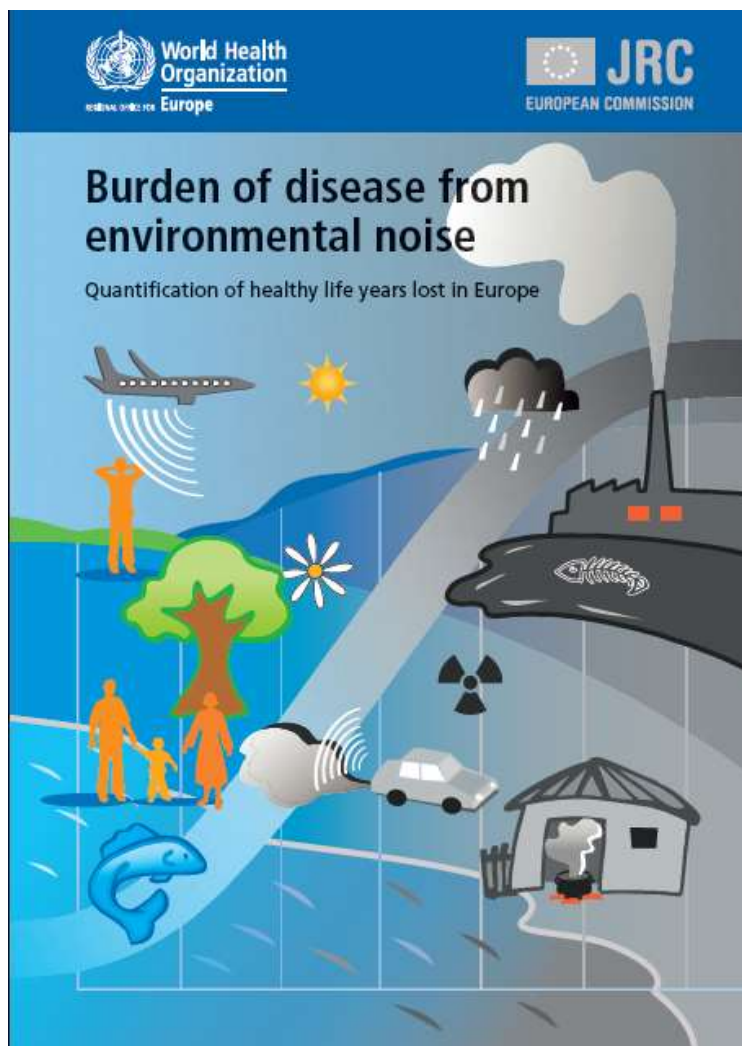


Open-plan offices:



- In a modern flexible OPO, the creation of a functional work station is a complex process in which acoustic planning is only one part of a series of considerations having to be addressed. The open-plan office should support both communication and concentrated work. Thus, for an OPO to be an efficient and comfortable place of work there are several other requirements than acoustic treatment that have to be fulfilled.

WHO report



Public health experts agree that environmental risks constitute 24% of the burden of disease. Widespread exposure to environmental noise from road, rail, airports and industrial sites contributes to this burden. One in three individuals is annoyed during the daytime and one in five has disturbed sleep at night because of traffic noise. **Epidemiological evidence indicates that those chronically exposed to high levels of environmental noise have an increased risk of cardiovascular diseases such as myocardial infarction.** Thus, noise pollution is considered not only an environmental nuisance but also a threat to public health.

Architectural acoustics



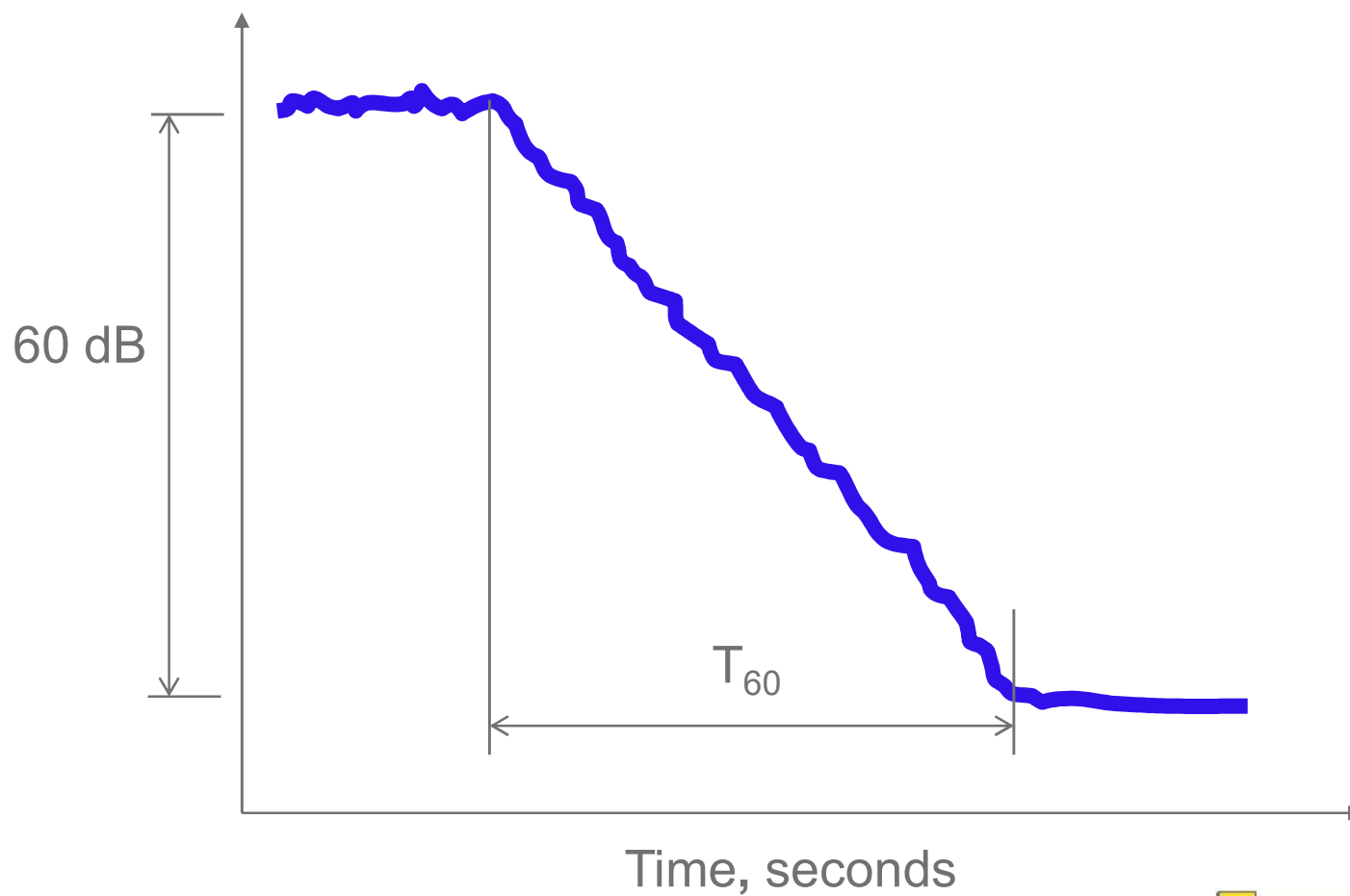
Wallace Clement Sabine

(June 13, 1868 – January 10, 1919)

American physicist who founded the field of architectural acoustics

Definition: Reverberation time

Sound pressure level, dB



His formula

$$T = 0.16 \left(\frac{V}{A} \right) \quad \text{or} \quad A = 0.16 \left(\frac{V}{T} \right)$$

where

T=the reverberation time (s)

V=the room volume (m³)

A=the total equivalent absorption area (m² sabin)

The equivalent absorption area A for a surface with area S m² is equal to $\alpha \times S$ where α is the absorption coefficient for the surface

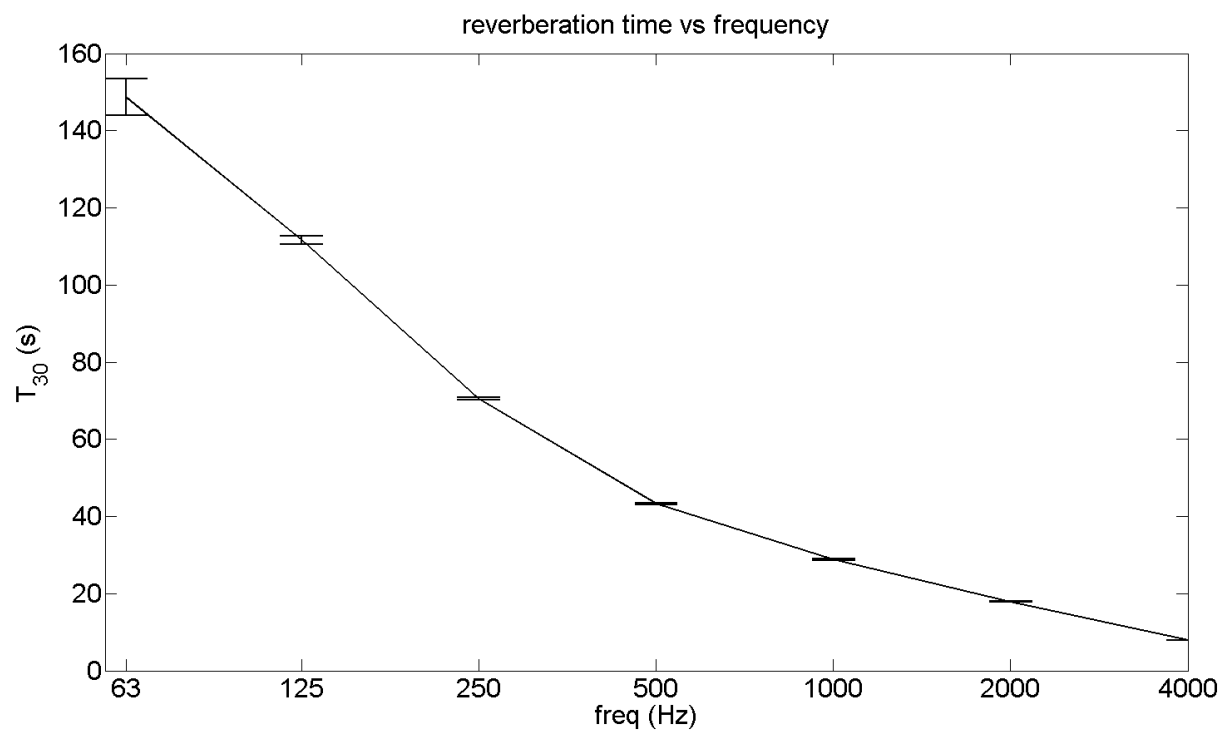


The oil-storage complex at Inchindown, near Invergordon in Scotland, was built during the Second World War

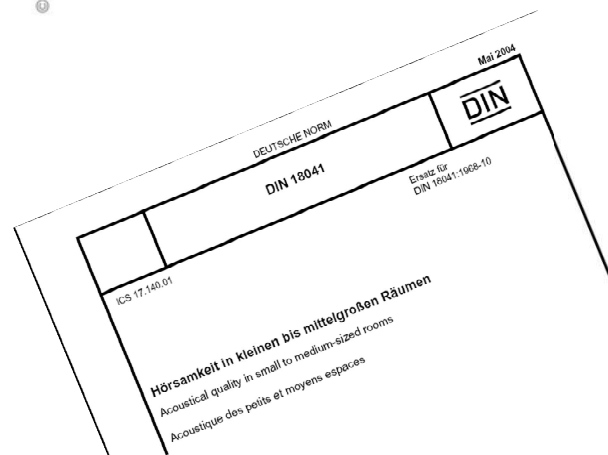
The broadband reverberation time, which considers all frequencies simultaneously, was 75 seconds – the figure certified as a world record by Guinness.

<https://soundcloud.com/tags/sonic%20wonderland>

The oil-storage complex at Inchindown



Standards and regulations



SVENSK STANDARD SS 25268:2007

Standarden gäller för byggnader med ett eller flera rum som är avsedda för verksamheter som kräver god ljudmiljö. Den gäller för alla typer av byggnader och rum, oavsett om de är nya eller befintliga. Den gäller för alla typer av byggnader och rum, oavsett om de är nya eller befintliga. Den gäller för alla typer av byggnader och rum, oavsett om de är nya eller befintliga.

Byggakustik – Ljudklassning av utrymmen i byggnader – Vårdlokaler, undervisningslokaler, dag- och fritidshem, kontor och hotell

Acoustics – Sound classification of spaces in buildings – Institutional premises, rooms for education, preschools and leisure-time centres, rooms for office work and hotels



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Commission de normalisation AFNOR S 30 D
Acoustique des lieux de travail

AFNOR S 30 D N 341

Date : 2005-09-08

Remplace le document N°

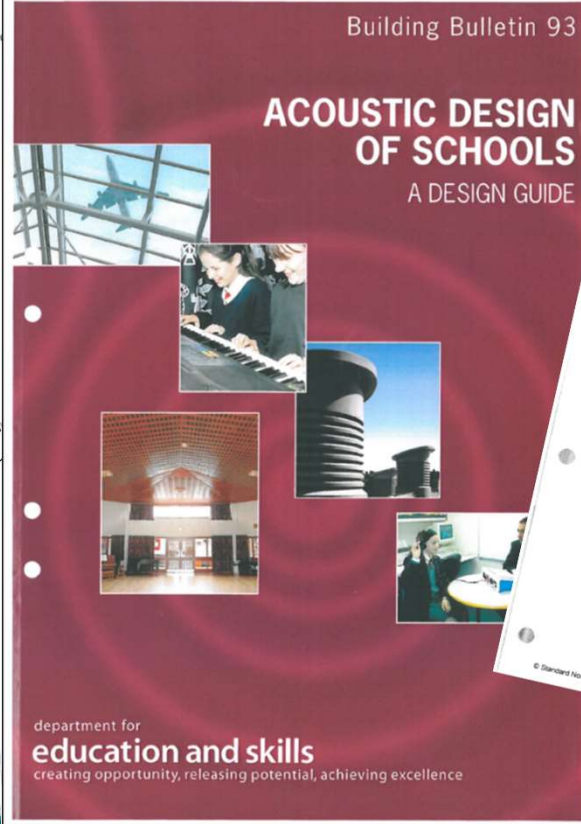
Projet de norme NF S 31-080 "Acoustique – Performances acoustiques des bureaux et espaces associés – Classification et critères de qualité des ambiances acoustiques par type d'espace" soumis à enquête

SUITE À DO

NOTE

SOURCE

FUSION



AMERICAN NATIONAL STANDARD

Acoustical Performance Criteria, Design Requirements, and Guidelines for Schools, 1: Permanent Schools

ANSI/ASA S12.60-2010/Part 1
(Revision of ANSI/ASA S12.60-2002)

Norsk Standard

NS 8175
2. utgave april 2005

ICS 91.120.20
Språk: Norsk

Lydforhold i bygninger
Lydklasser for ulike bygningstyper
Acoustic conditions in buildings
Sound classification of various types of buildings

© Standard Norge. Henvendelse om gjengivelse rettes til Proform AS, www.standard.no

Ecophon®
SAINT-GOBAIN

A SOUND EFFECT ON PEOPLE

Acoustic design with Sabine formula

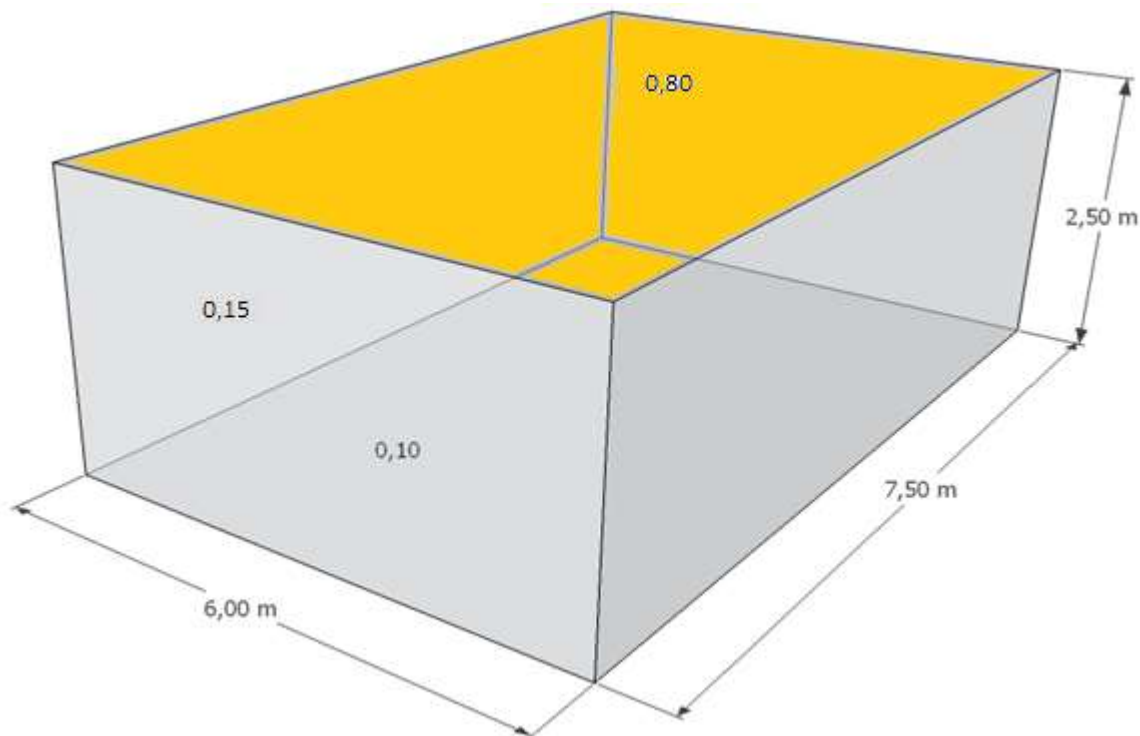
Example: The reverberation time in a room with a volume of 200 m³ is 2,5 s at 1000 Hz.

Target value for the reverberation time is 0,40 s at 1000 Hz

- ❑ $A(\text{before treatment}) = 0,161 \times V / T = 0,161 \times 200 / 2,5 = 12,9 \text{ m}^2 \text{ sabin}$
- ❑ $A(\text{needed to fulfil } 0,40 \text{ s}) = 0,161 \times V / T = 0,161 \times 200 / 0,40 = 80,5 \text{ m}^2 \text{ sabin}$
- ❑ $A(\text{ to be added to fulfil } 0,40 \text{ s}) = A(\text{needed}) - A(\text{before}) = 80,5 - 12,9 = 67,6 \text{ m}^2 \text{ sabin}$

If e.g. the absorption coefficient for a ceiling absorber is 0,90 at 1000 Hz we will need $S = A / \alpha = 67,6 / 0,90 = 75 \text{ m}^2$

Sabine formula: How it works in theory



Absorption coefficients (500 Hz):

Walls=0,15

Ceiling=0,80

Floor=0,10

Absorption data from EN 12354-6

Table B.1 — Typical values for the absorption coefficient

Material	Sound absorption coefficient α_s in octave bands, centre frequency in Hz					
	125	250	500	1 000	2 000	4 000
concrete, plastered brick	0,01	0,01	0,01	0,02	0,02	0,03
brickwork, unplastered	0,02	0,02	0,03	0,04	0,05	0,07
hard floor coverings (e.g. PVC, parquet) on heavy floor	0,02	0,03	0,04	0,05	0,05	0,06
soft floor covering on heavy floor; ≤ 5 mm	0,02	0,03	0,06	0,15	0,30	0,40
soft floor covering on heavy floor; ≥ 10 mm	0,04	0,08	0,15	0,30	0,45	0,55
wooden floor, parquet on battens	0,12	0,10	0,06	0,05	0,05	0,06
windows, glass facade	0,12	0,08	0,05	0,04	0,03	0,02
doors (wood)	0,14	0,10	0,08	0,08	0,08	0,08
net curtain; 0 mm - 200 mm in front of hard surface ¹	0,05	0,04	0,03	0,02	0,02	0,02
curtain, $< 0,2$ kg/m ² ; 0 mm – 200 mm in front of hard surface; typical minimum ¹	0,05	0,06	0,09	0,12	0,18	0,22
curtain, woven material $\approx 0,4$ kg/m ² ; folded or ruffled $> 1:3$, 0-200 mm in front of hard surface; typical maximum	0,10	0,40	0,70	0,90	0,95	1,00
large openings (smallest dimension > 1 m)	1,00	1,00	1,00	1,00	1,00	1,00
air grid, 50 % open area	0,30	0,50	0,50	0,50	0,50	0,50
NOTE These data are based on publications used in Austria, Denmark and the Netherlands.						
¹ in front of a window the values of the combination can increase to the values for such a window alone.						

Absorption data from EN 12354-6

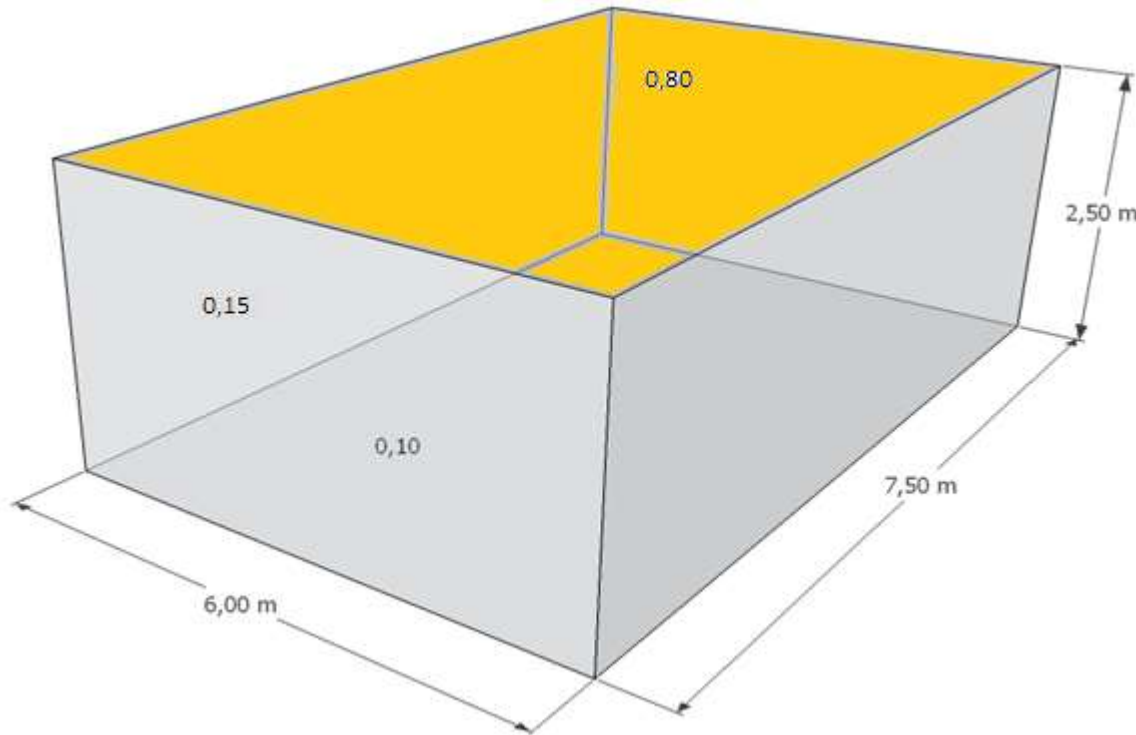
Table C.1 — Typical values for the equivalent absorption area for some common objects

Object	Equivalent absorption area A_{obj} in octave bands, centre frequency in Hz					
	125	250	500	1 000	2 000	4 000
single chair, wood	0,02	0,02	0,03	0,04	0,04	0,04
single chair, upholstered	0,10	0,20	0,25	0,30	0,35	0,35
single person in a group, sitting or standing, 1 per 6 m ² area; typical minimum	0,05	0,10	0,20	0,35	0,50	0,65
single person in a group, sitting, 1 per 6 m ² area; typical maximum	0,12	0,45	0,80	0,90	0,95	1,00
single person in a group, standing, 1 per 6 m ² area; typical maximum	0,12	0,45	0,80	1,20	1,30	1,40
NOTE These data are based on publications used in Austria, Denmark and the Netherlands.						

Table C.2 — Typical vales for the sound absorption coefficient for some common specified arrays of objects

Array of objects	Sound absorption coefficient α_s in octave bands, centre frequency in Hz					
	125	250	500	1 000	2 000	4 000
chairs in a row at 0,9m – 1,2m; wood/plastic	0,06	0,08	0,10	0,12	0,14	0,16
chairs in a row at 0,9m – 1,2m; upholstered; typical minimum	0,10	0,20	0,30	0,40	0,50	0,50
chairs in a row at 0,9m – 1,2m; upholstered; typical maximum	0,50	0,70	0,80	0,90	1,0	1,0
persons sitting in a row at 0,9m – 1,2m (audience); typical minimum	0,20	0,40	0,50	0,60	0,70	0,70
persons sitting in a row at 0,9m – 1,2m (audience); typical maximum	0,60	0,70	0,80	0,90	0,90	0,90
children in a hard furnished class room, 1 per m ² area	0,10	0,20	0,25	0,35	0,40	0,40
NOTE These data are based on publications used in Austria, Denmark and the Netherlands.						

Sabine formula: How it works in theory



Absorption coefficients (500 Hz):

Walls=0,15

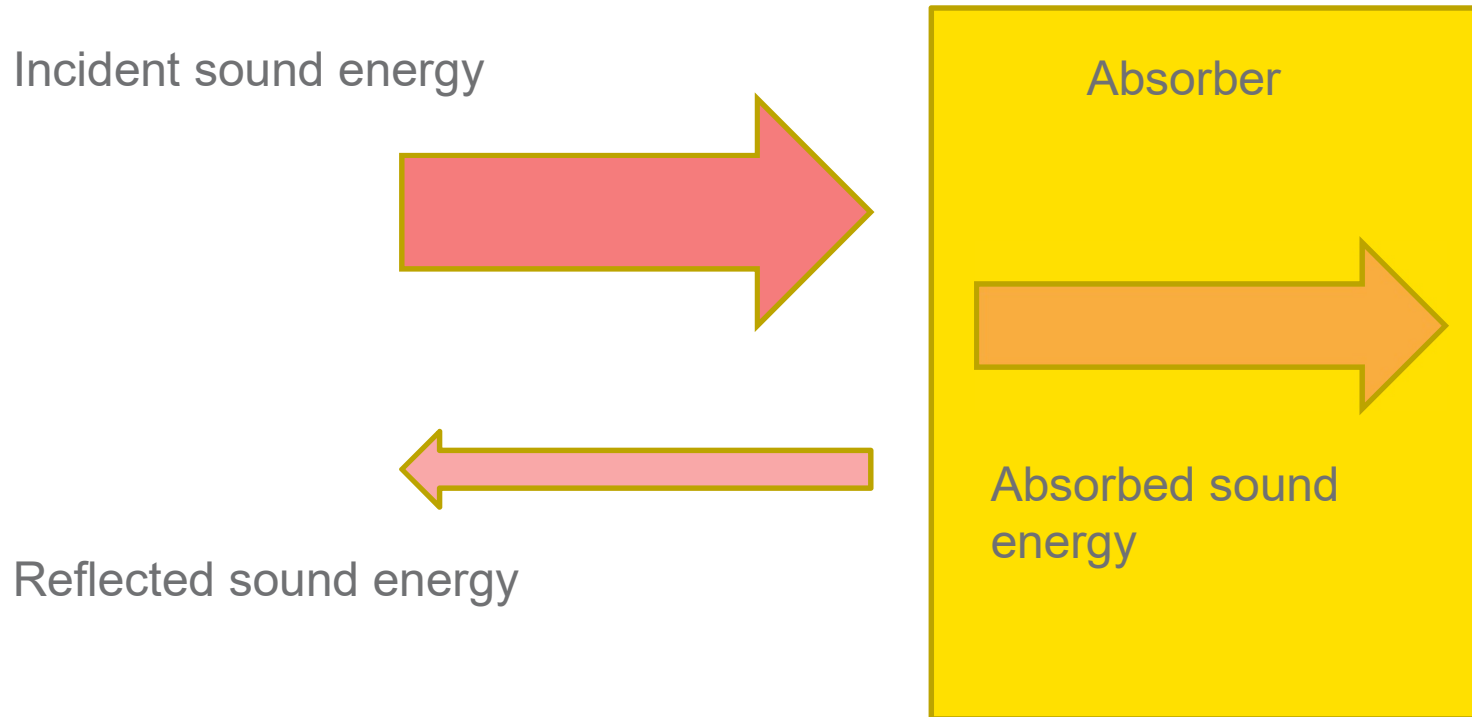
Ceiling=0,80

Floor=0,10

$$A = \sum \alpha_i \times S_i = 0,10 \times 6 \times 7,5 + 2 \times 0,15 \times 7,5 \times 2,5 + 2 \times 0,15 \times 6 \times 2,5 + 0,80 \times 6 \times 7,5 = 51 \text{ m}^2 \text{ sabin}$$

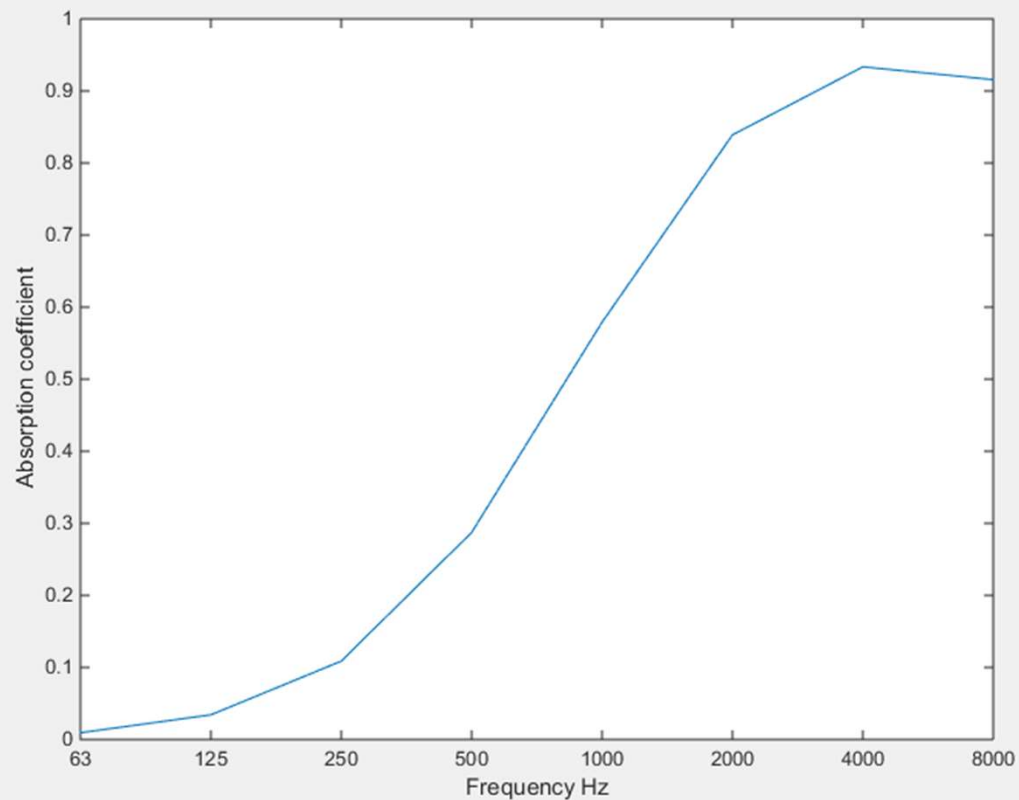
$$T_{60} = 0,161 \times (V/A) = 0,161 \times 112,5 / 51 \approx 0,36 \text{ s}$$

Absorption coefficient at normal incidence



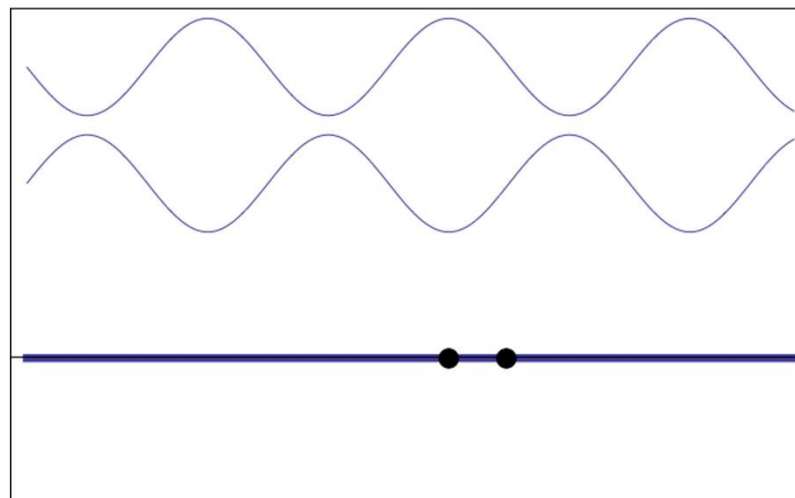
$$\text{Absorption coefficient } \alpha = \frac{\text{Absorbed sound energy}}{\text{Incident sound energy}}$$

Absorption coefficient as a function of frequency



20 mm glass
wool absorber
directly
mounted in
front of a hard
surface
(concrete)

Standing wave

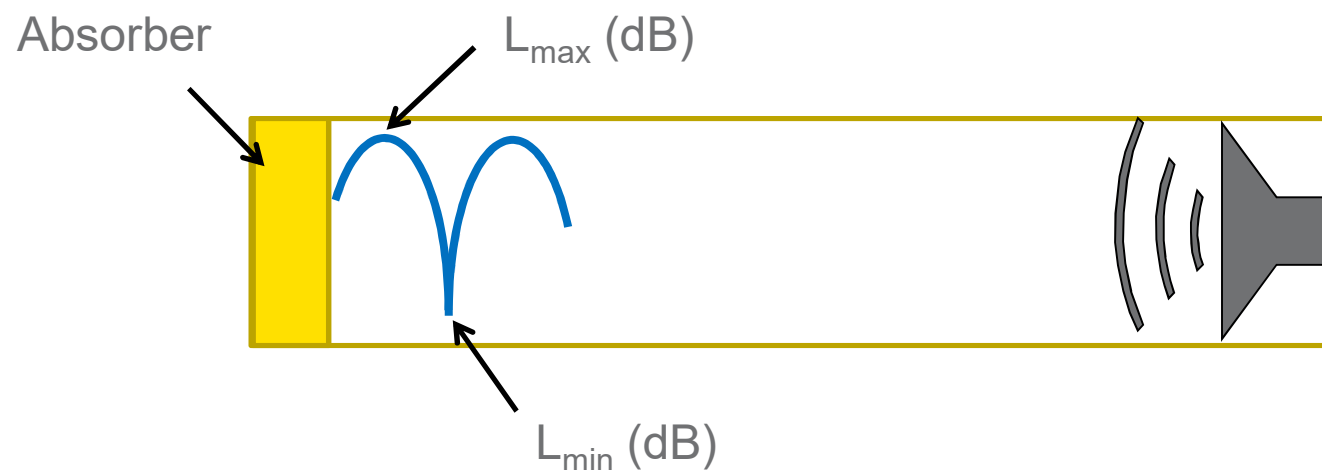


wave 1 going right

wave 2 going left

wave 1 + wave 2

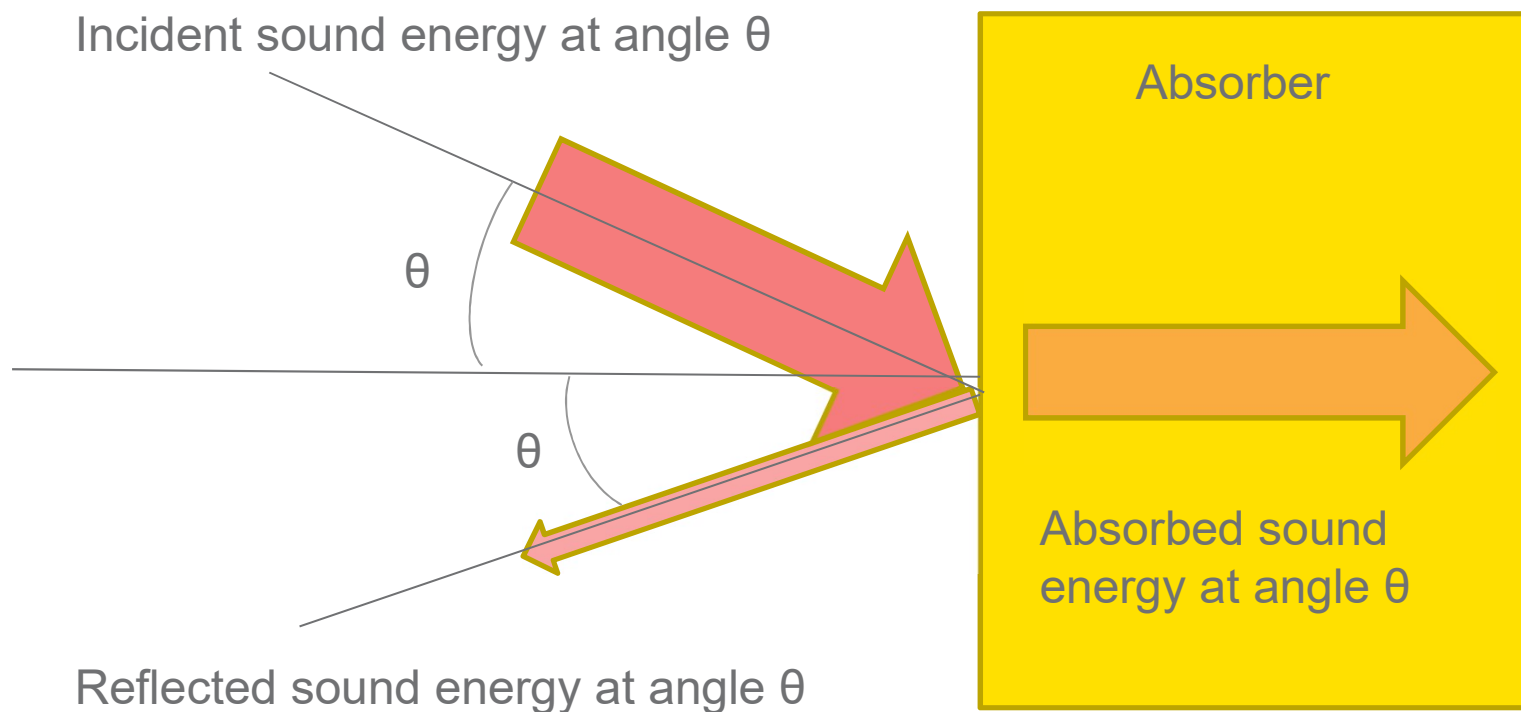
Impedance tube (Kundt's tube)



$$F = 10^{(L_{max} - L_{min})/20}$$

$$\alpha = \frac{4F}{(1 + F)^2}$$

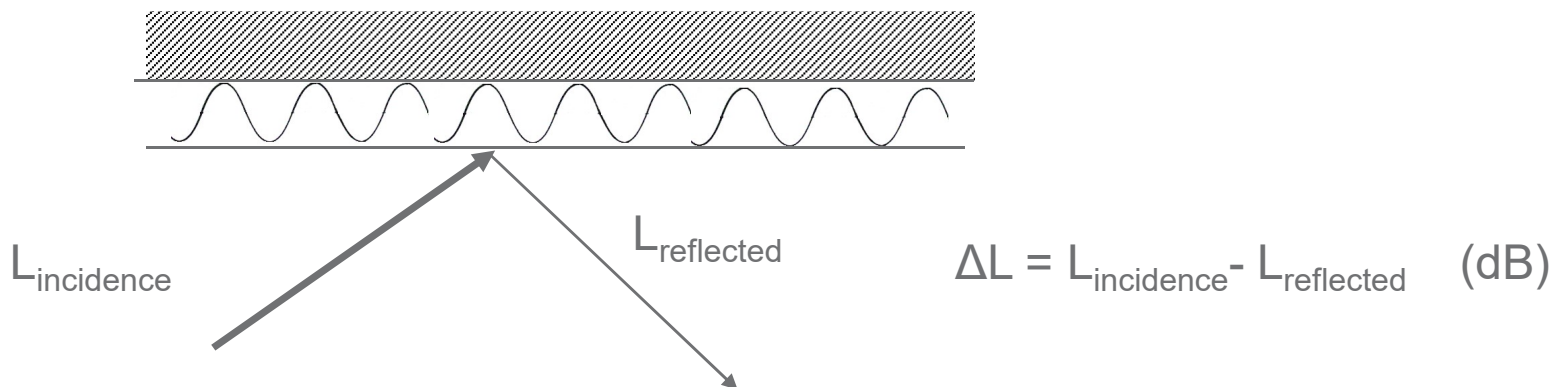
Absorption coefficient



$$\text{Absorption coefficient } \alpha(\theta) = \frac{\text{Absorbed sound energy at angle } \theta}{\text{Incident sound energy at angle } \theta}$$

Reflected sound

$$\Delta L = 10 \cdot \log(1 - \alpha)$$



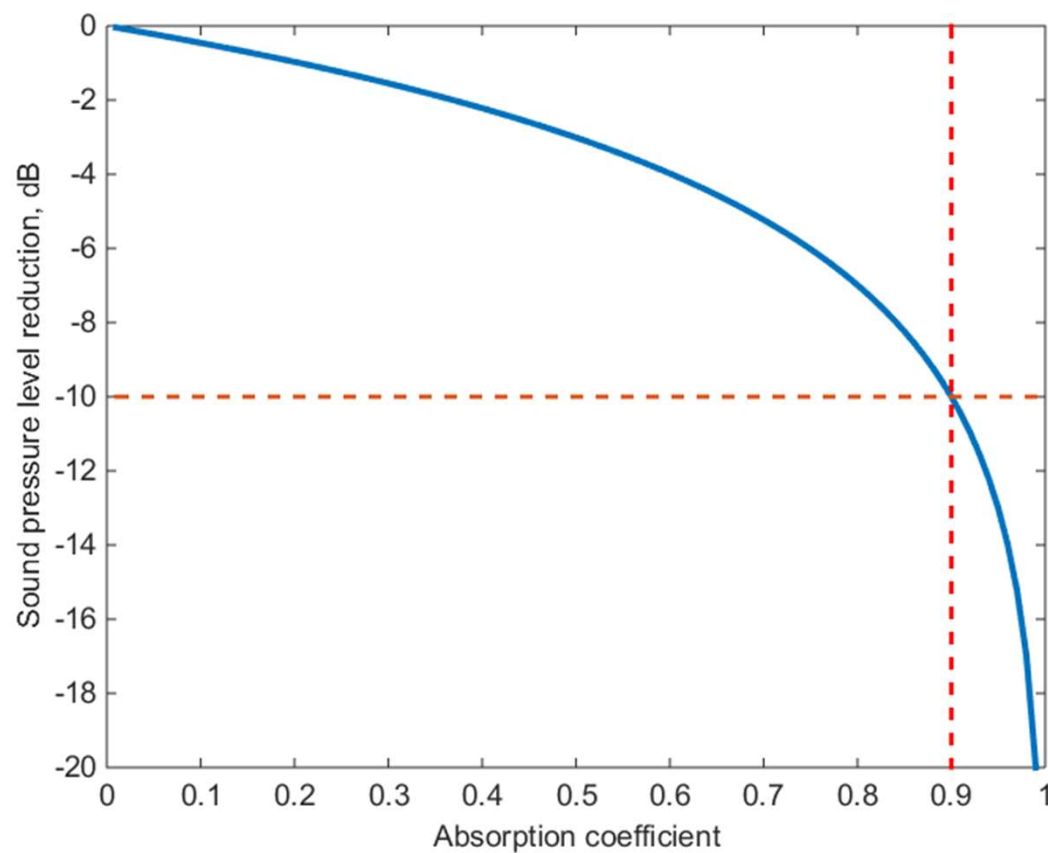
**Absorption
coefficient**

0,30 – 0,55 (class D)
0,60 – 0,75 (class C)
0,80 – 0,85 (class B)
0,90 – 0,99 (class A)

**The reflected sound is
reduced by**

1-4 dB
4-7 dB
7-10 dB
10-20 dB

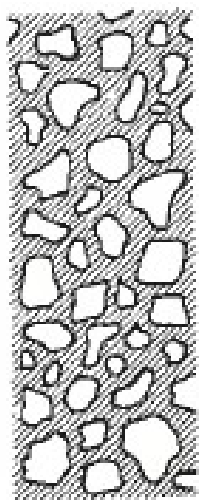
Level reduction of sound reflection



Glass wool



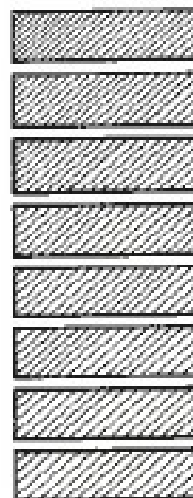
Open and closed structures



Closed cell

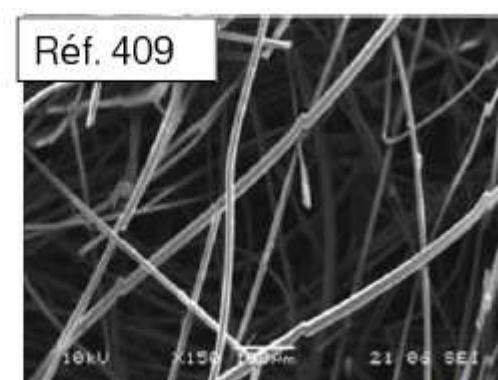
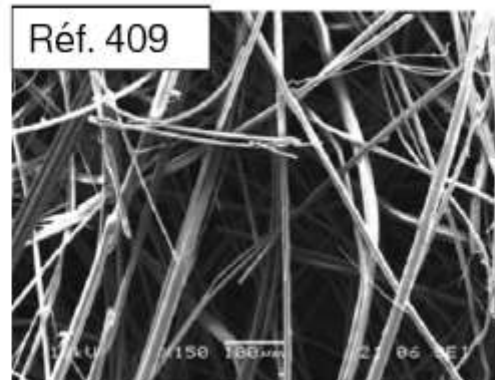
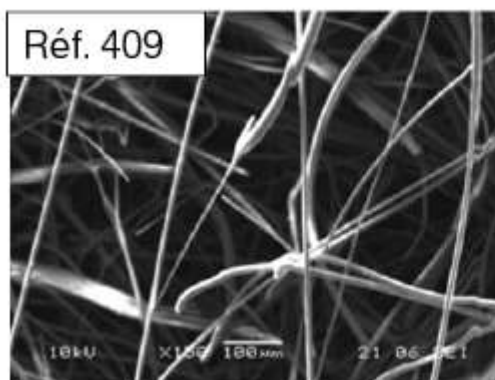
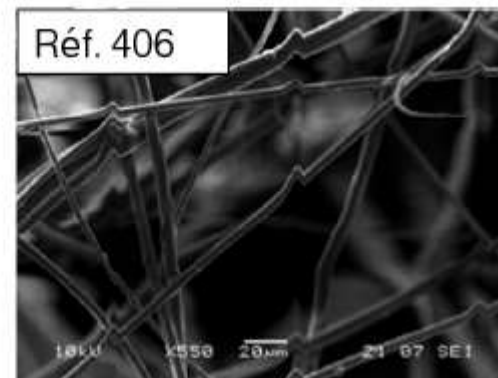
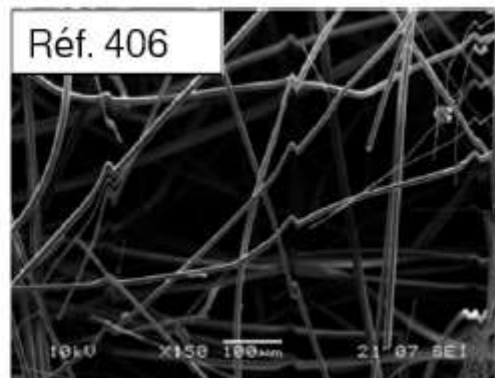
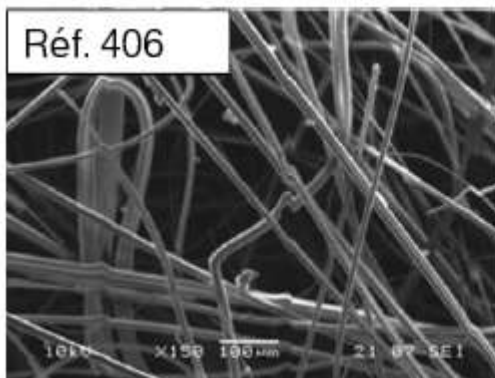


Open cell



Simple model
of porous
absorber

Microscopic structure



Acoustical behavior of porous materials

Porous materials {
Solid phase
Fluid phase

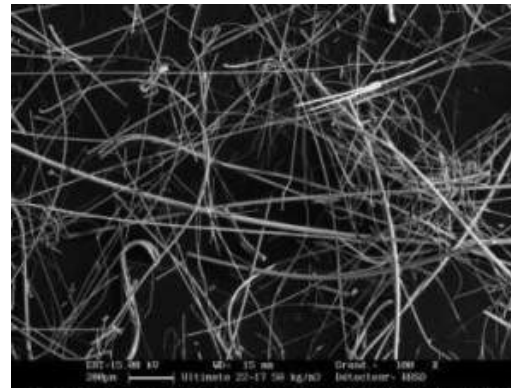
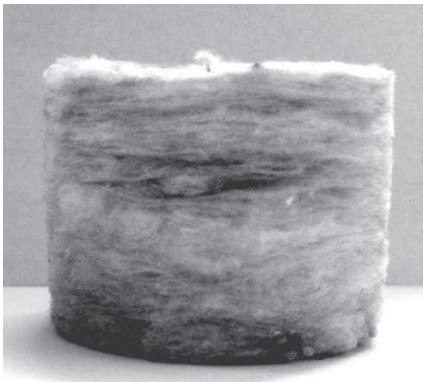


Porous materials are used to
dissipate energy:

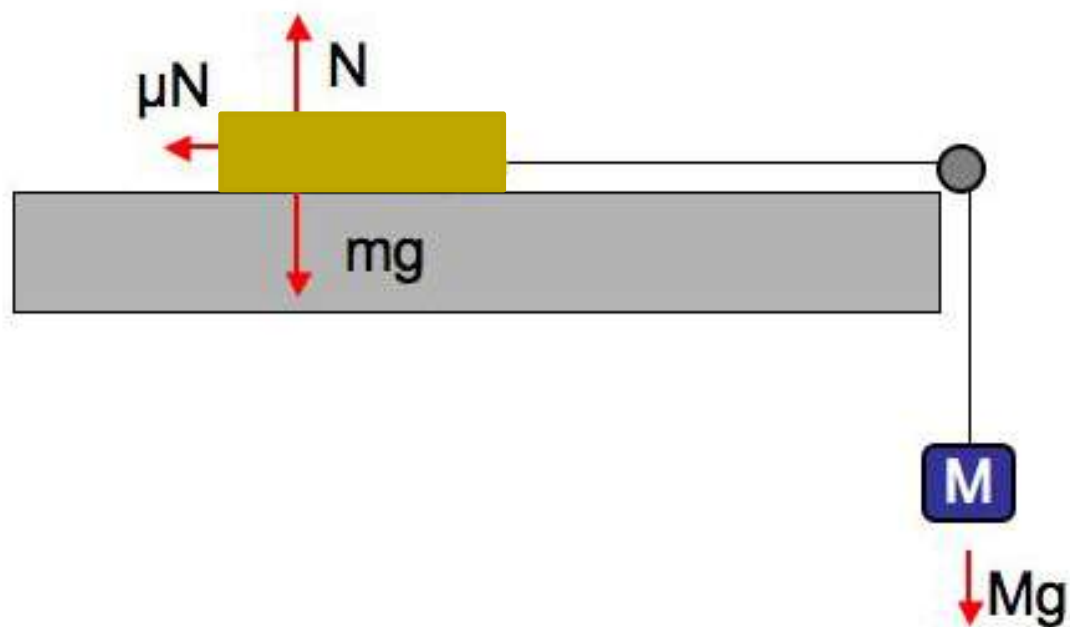
Acoustic energy is being
transferred into heat

General properties:

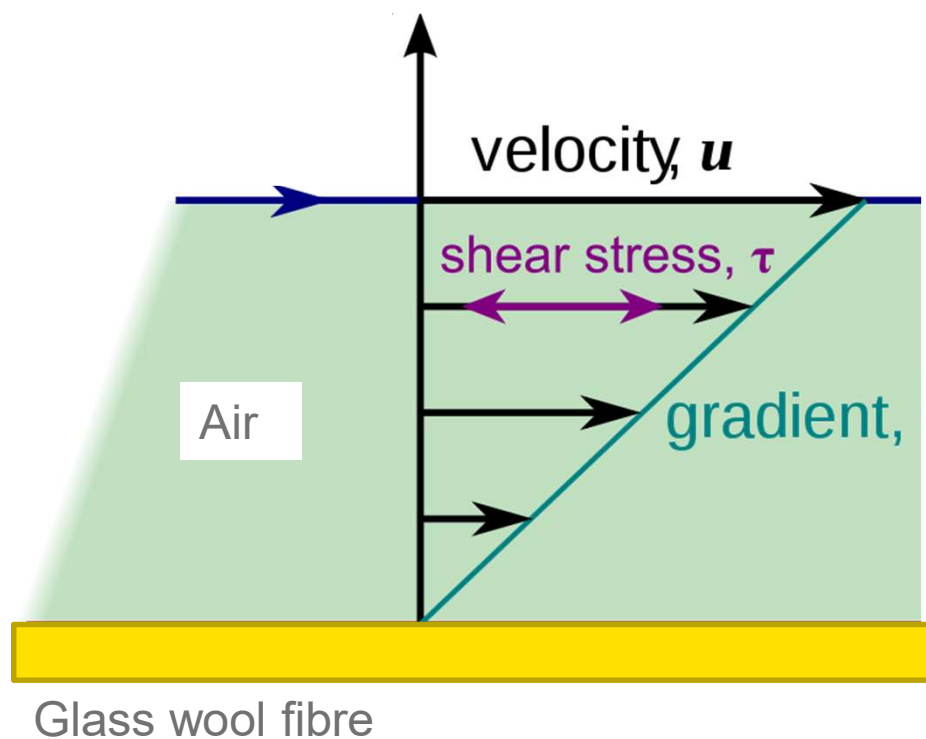
- Characteristic pore size between μm and mm
- Open porosity



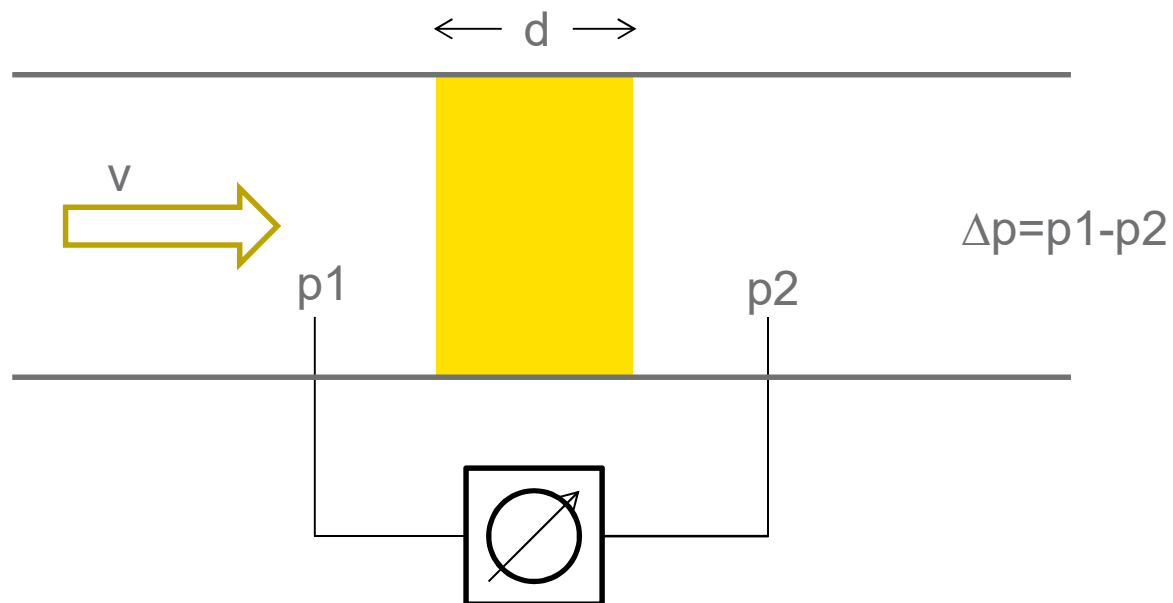
Friction



Viscosity: energy converted into heat (friction losses)

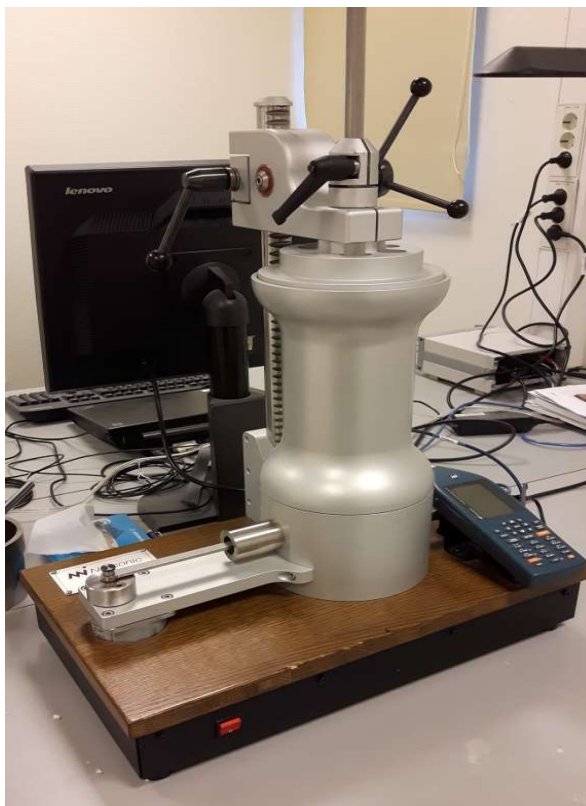


Measurement of air flow resistivity (ISO 9053)



$$\text{Flow resistivity } r = \frac{\Delta p}{v \cdot d} \quad \left(\frac{\text{Pa} \cdot \text{s}}{\text{m}^2} \right)$$

Air flow resistance equipment



Acoustical behavior of porous materials

- Physical phenomenon:
 - Viscous and thermal exchanges
 - When acoustical wave is propagating in the porous materials, **frictional losses** and **heat exchanges** between solid and fluid phases occur.
 - Structural damping
 - Energy dissipation by deformation of the structure

Models for characterizing porous material

- Delany and Bazley's model (1970)
- Miki's model (1990)
- Komatsu's model (2008)

Models for characterizing porous material

- Delany and Bazley's model (1970)
- Miki's model (1990)

Characteristic impedance

$$Z_c = \rho_0 c_0 \left[1 + 0.070 \left(\frac{f}{\sigma} \right)^{-0.632} - j 0.107 \left(\frac{f}{\sigma} \right)^{-0.632} \right]$$

Airflow resistivity

- Komatsu's model (2008)

Acoustical behavior of porous materials

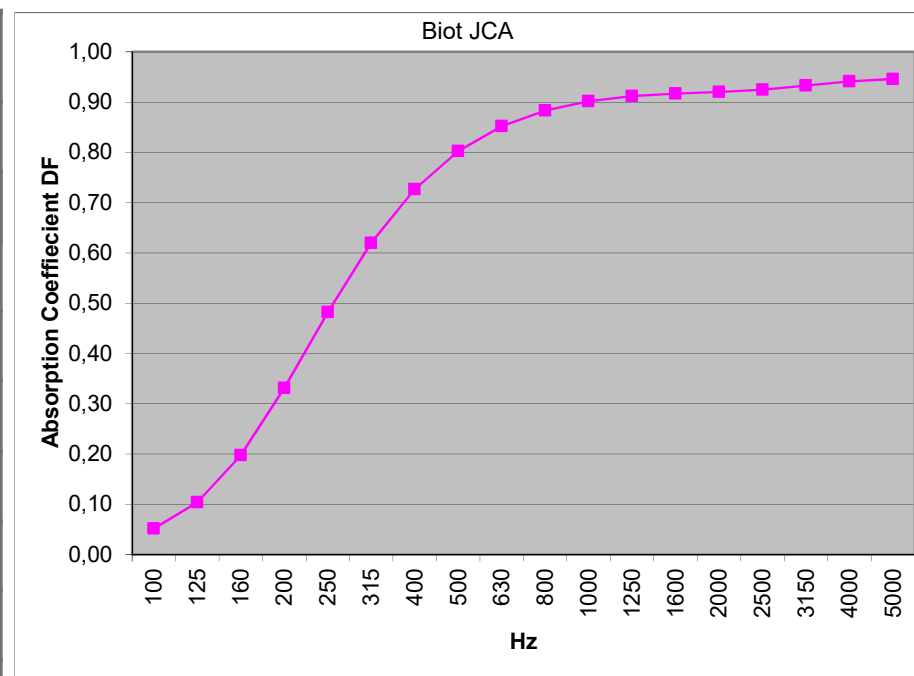
Sound propagation in
poro-elastic media
(Biot-Jonhson-Champoux- Allard
model)

• Fluid density	ρ_f
• Flow resistivity	σ
• Open porosity	ϕ
• Tortuosity	α_∞
• Viscous characteristic length	Λ
• Thermal characteristic length	Λ'
• Solid density	ρ_s
• Young's modulus	E
• Shear modulus	N

Parametric Study: materials properties and sound absorption

– Properties of Glass wool

Properties	Glass wool
Thickness (mm)	50
σ , airflow resistivity (Pa s/m ²)	35000
Φ , Porosity	0.96
α_{∞} , Tortuosity	1.02
Λ , Viscous Characteristic Length (μm)	52
Λ' , Thermal Characteristic Length (μm)	61
ρ , Density (kg/m ³)	40
E, Young Modulus (Pa)	4000
η , Loss factor	0.31
N, Poisson Ratio	0



Acoustical behavior of porous materials

- Perspectives
 - Modify the microstructure of the materials
 - Length of the fibers
 - Diameter of the fibers
 - Space between fibers
 - Arrangement of the fibers
 -

Flow resistivity as a function of density

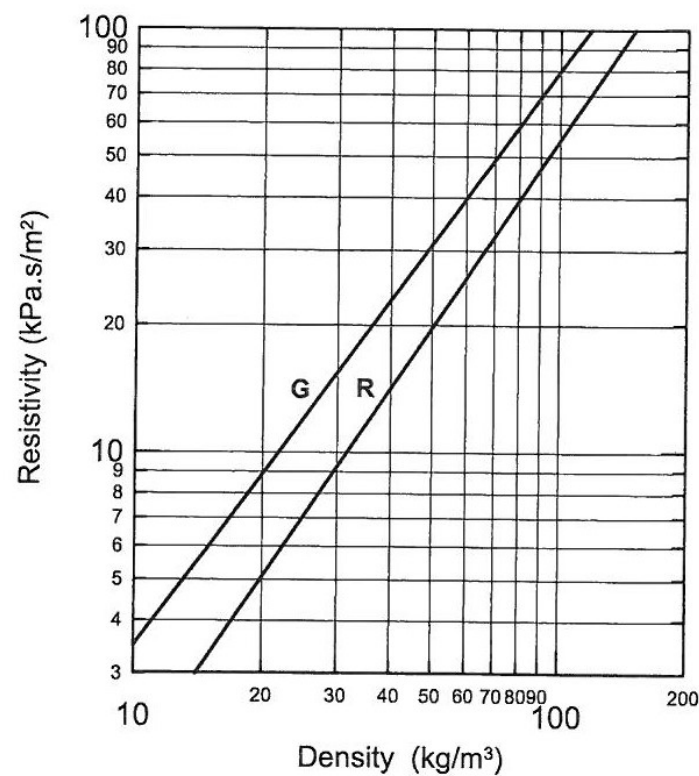
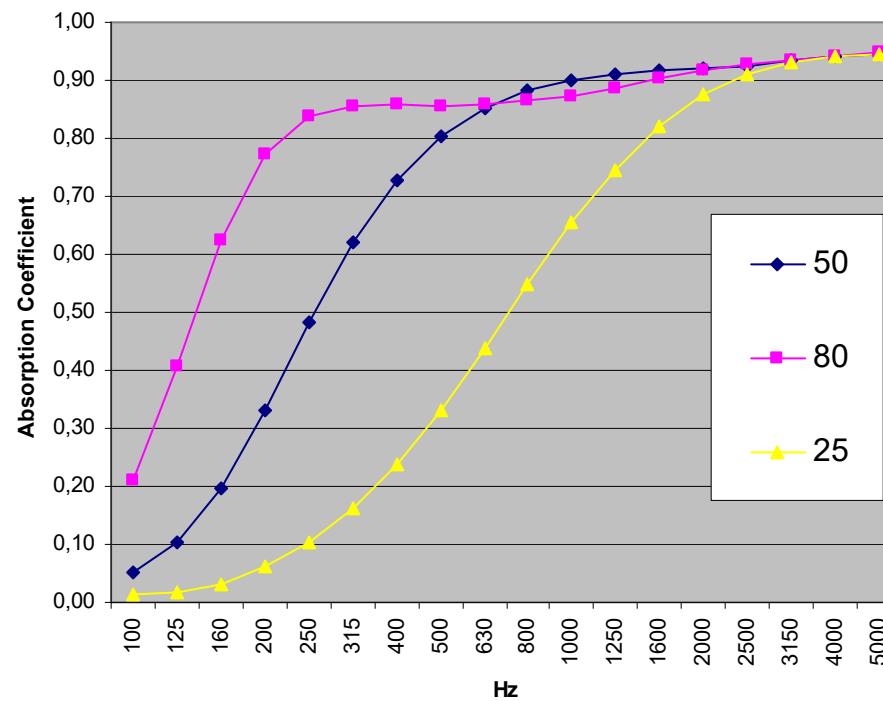


Figure 5.33 Typical data for airflow resistivity of mineral wool, glass wool (G) and rock wool (R) as a function of density.

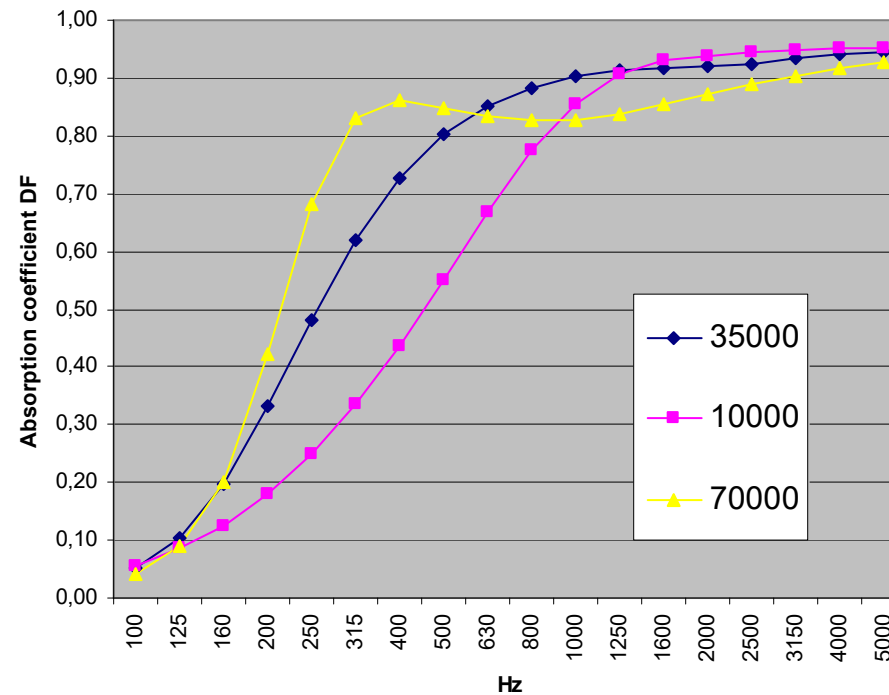
Parametric Study: materials properties and sound absorption

- Parametric Study
 - Influence of the thickness of the porous material
 - 50mm, 25mm and 80mm



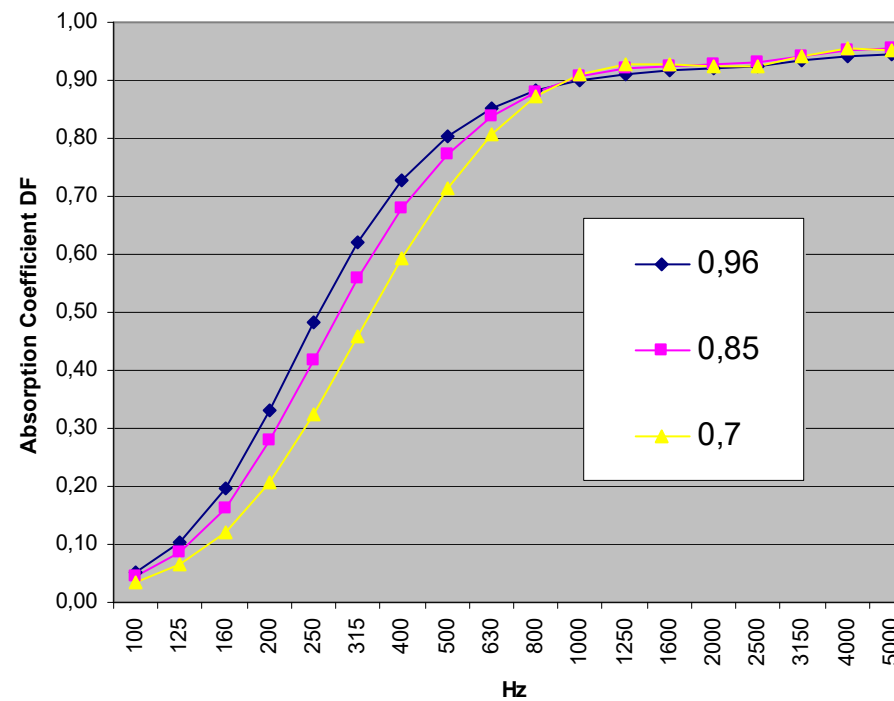
Parametric Study: materials properties and sound absorption

- Parametric Study
 - Influence of the flow resistivity
 - 35000N.s.m⁻⁴, 10000N.s.m⁻⁴ and 70000N.s.m⁻⁴



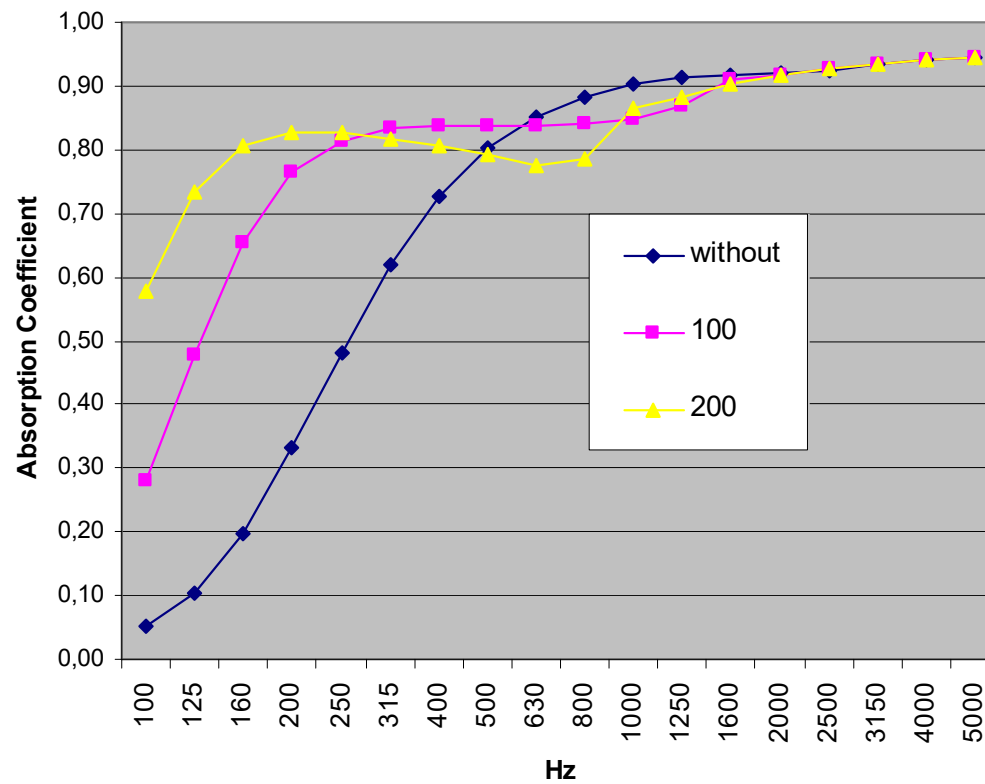
Parametric Study: materials properties and sound absorption

- Parametric Study
 - Influence of the porosity
 - 0.96, 0.85 and 0.7



Parametric Study: materials properties and sound absorption

- Parametric Study
 - Influence of air cavity
 - 100mm and 200mm



Absorbers

1. Porous absorbers

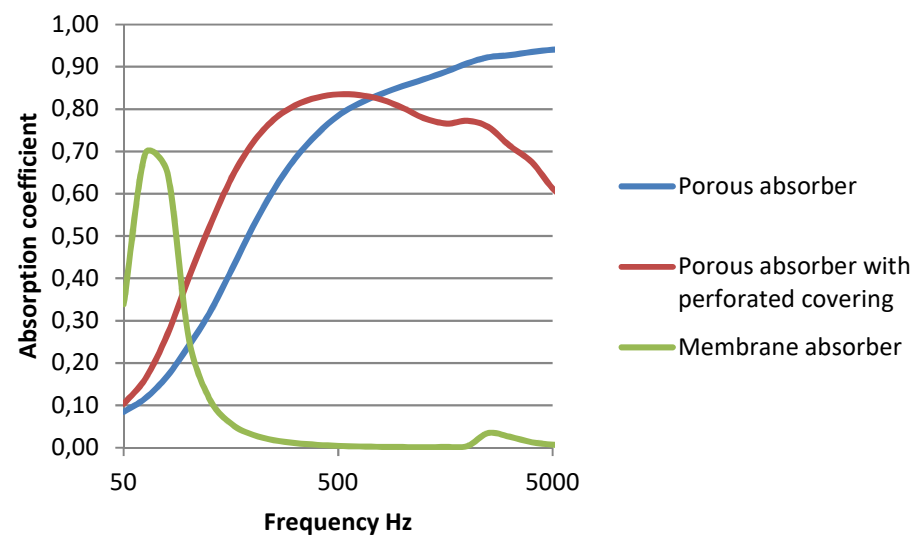
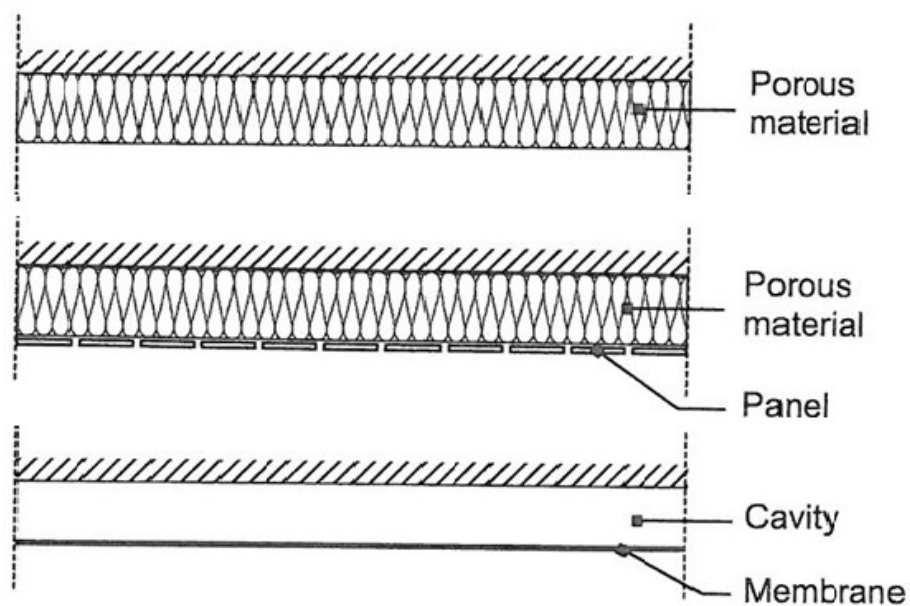
(mineral wool products, porous fibreboard products, foam plastic, fabric, felt etc)

2. Resonance absorbers

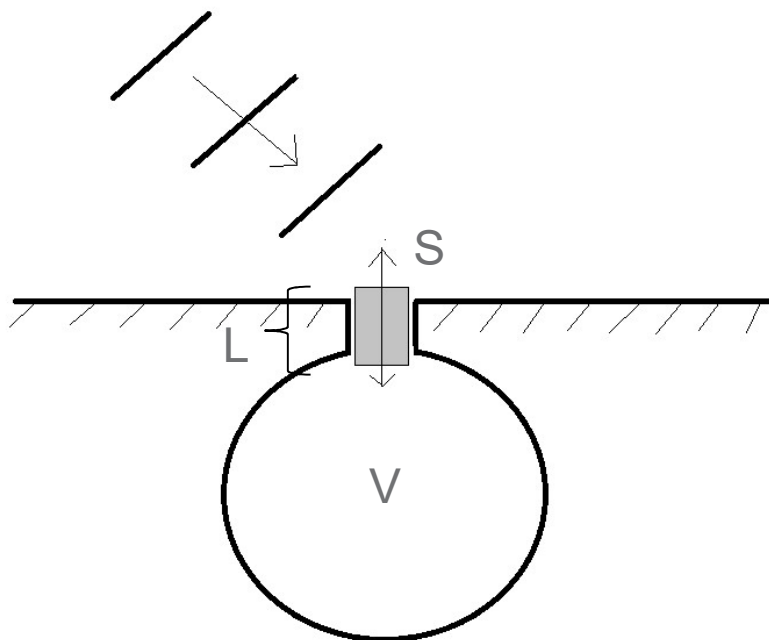
a) Cavity absorbers (Helmholtz absorbers)

b) Membrane absorbers

Different types of absorbers



Helmholtz resonator



Resonance frequency (Hz)

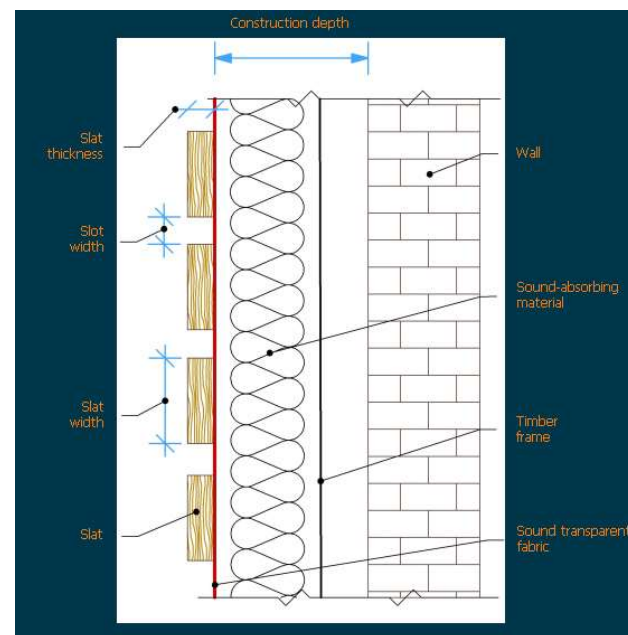
$$f_r = 55 \cdot \sqrt{\frac{S}{L \cdot V}}$$

S = the cross sectional area of the neck
 L =approx. the length of the neck
 V = the volume of the cavity

Example: “an empty bottle of wine”, $S=2,54 \times 10^{-4} \text{ m}^2$,
 $L= 0,08 \text{ m}$, $V= 0,75 \times 10^{-3} \text{ m}^3$

$f_r = 113 \text{ Hz}$

Slotted absorber



$$f_0 = \frac{c}{2\pi} \sqrt{\frac{r}{d \cdot 1.2 \cdot D \cdot (r + w)}}$$

w - slat width,
r - slot width,
d - slat thickness,
D - frame depth,
c - speed of sound in the air.

Standards

ISO 354:2003 Acoustics – Measurements of sound absorption in a reverberation room

- Plane absorbers
- Discrete sound absorbers
- Baffles

ISO 11654:1997 Acoustics – Sound absorbers for use in buildings – Rating of sound absorption

- Practical absorption coefficient α_p
- Weighted sound absorption coefficient α_w
- Sound absorption classes A to E

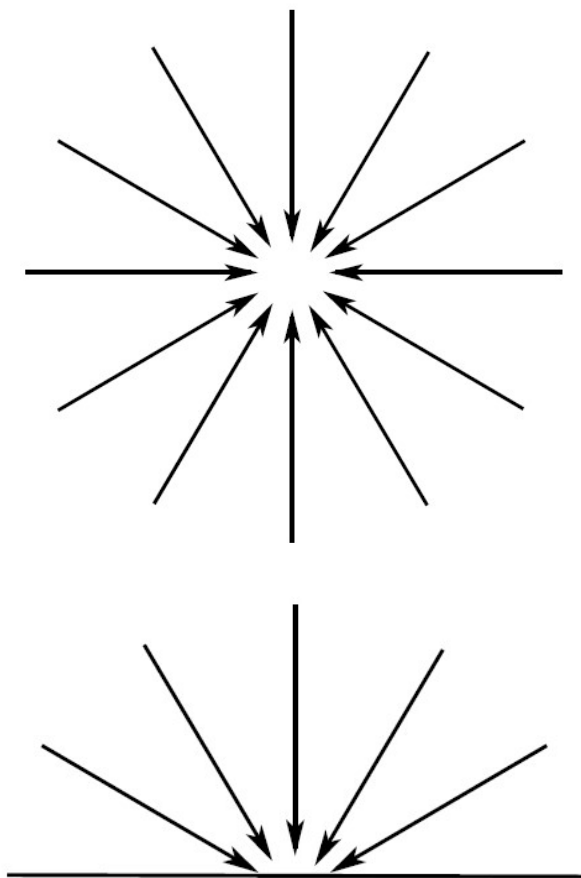
US standard: ASTM C423, “Sound absorption and sound absorption coefficients by the reverberation room method”

- Recommended room volume: 200 m³ (ISO 354 recommends 200 m³)
- Recommended test area: 6,69 m² (ISO 354 recommends 10 to 12 m²)
- Typical ods.: 400 mm (ISO 354: typical ods. 200 mm)

Absorption coefficients (single number):

- **Sound absorption average, SAA:** average of the sound absorption coefficients for the twelve one-third octave bands from 200 Hz to 2500 Hz.
- **Noise reduction coefficients, NRC:** average for the sound absorption coefficients for 250, 500, 1000, and 2000 Hz rounded to the nearest multiple of 0.05.

Diffuse sound field



Echo chamber of the Dresden University of Technology

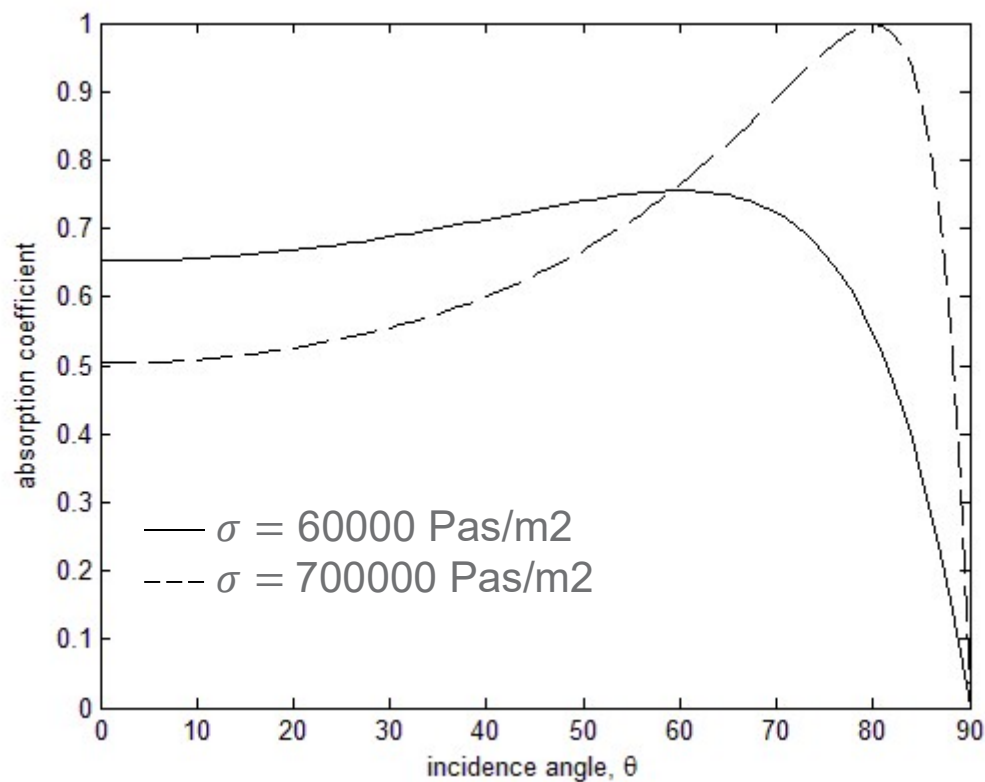
On the use of practical absorption coefficients



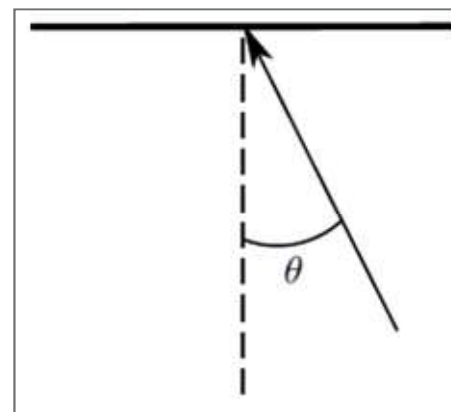
On the use of practical absorption coefficients



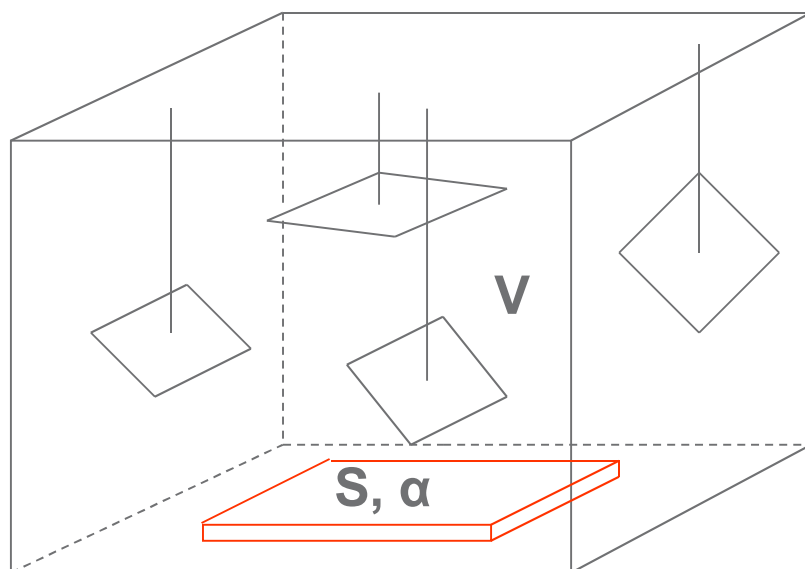
Angle dependent absorption coefficient



— $\sigma = 60000 \text{ Pas/m}^2$
--- $\sigma = 700000 \text{ Pas/m}^2$



Measurement of absorption coefficients according to EN ISO 354



$$\Delta A = A_{with} - A_{without} = \frac{0.16 \cdot V}{T_{with}} - \frac{0.16 \cdot V}{T_{without}}$$

A=equivalent absorption area, m² sabine

V= room volume, m³

T= reverberation time, s

absorption coefficient

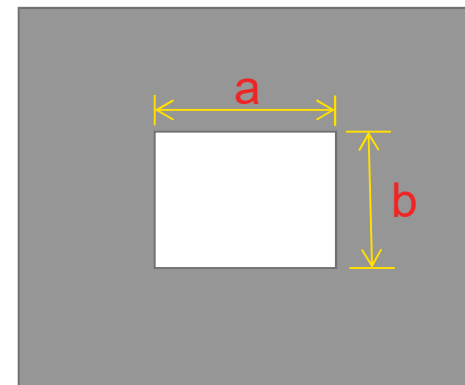
$$\alpha = \Delta A / S$$

Equivalent absorption area A (m² sabin)

$$T = 0.16 \left(\frac{V}{A} \right)$$

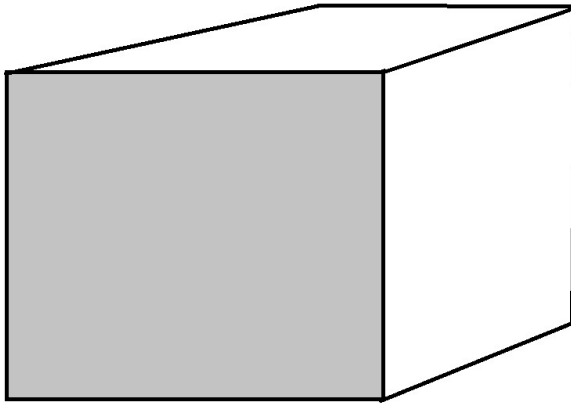
$$A = \alpha \cdot S$$

if $\alpha=1$ then $A=S$

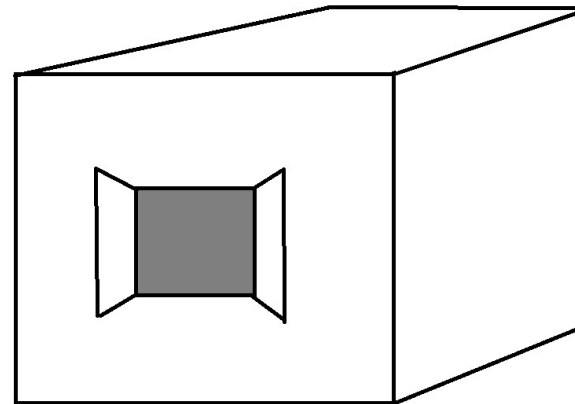


“Open window”
 $A=S=a \times b$

Example: Equivalent absorption area



Wall panel
 $\alpha=0.3$
 $S=10 \text{ m}^2$
 $A=0.3 \times 10=3 \text{ m}^2 \text{ sabin}$



Open window
 $\alpha=1.0$
 $S=3 \text{ m}^2$
 $A=1.0 \times 3=3 \text{ m}^2 \text{ sabin}$

The open window with an area of 3 m^2 absorbs the same amount of sound power as the wall panel with an alpha of 0.3 and an area of 10 m^2

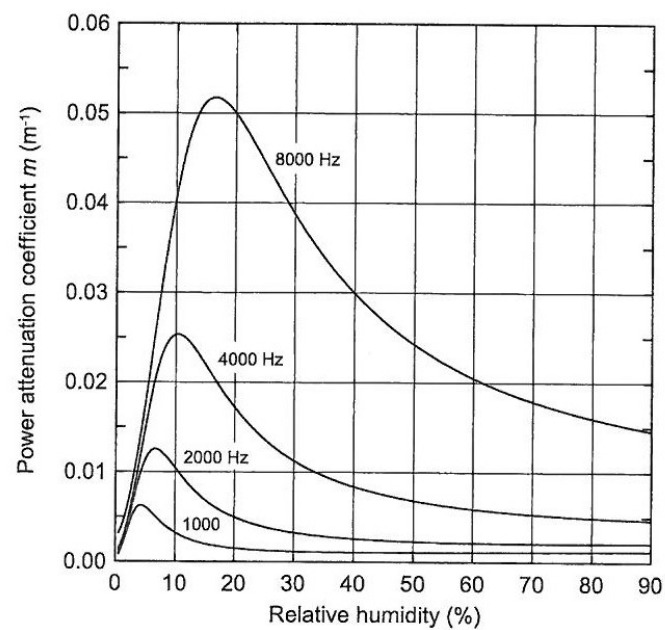
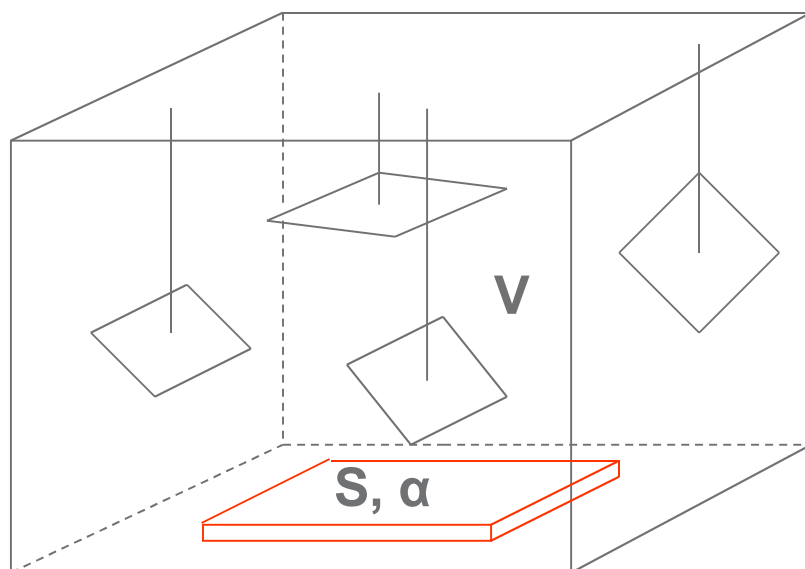


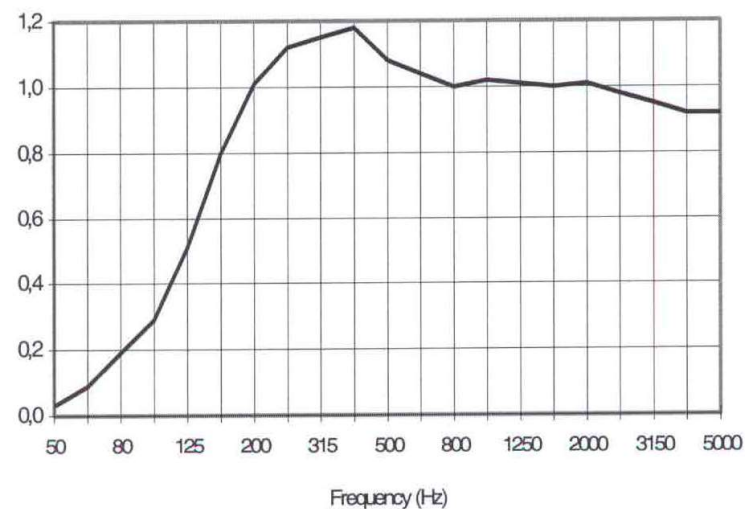
Figure 4.8 Power attenuation coefficient m for atmospheric absorption at 20° Celsius. Calculated from ISO 9613-1.

Measurement of sound absorption in reverberation room (ISO 354)



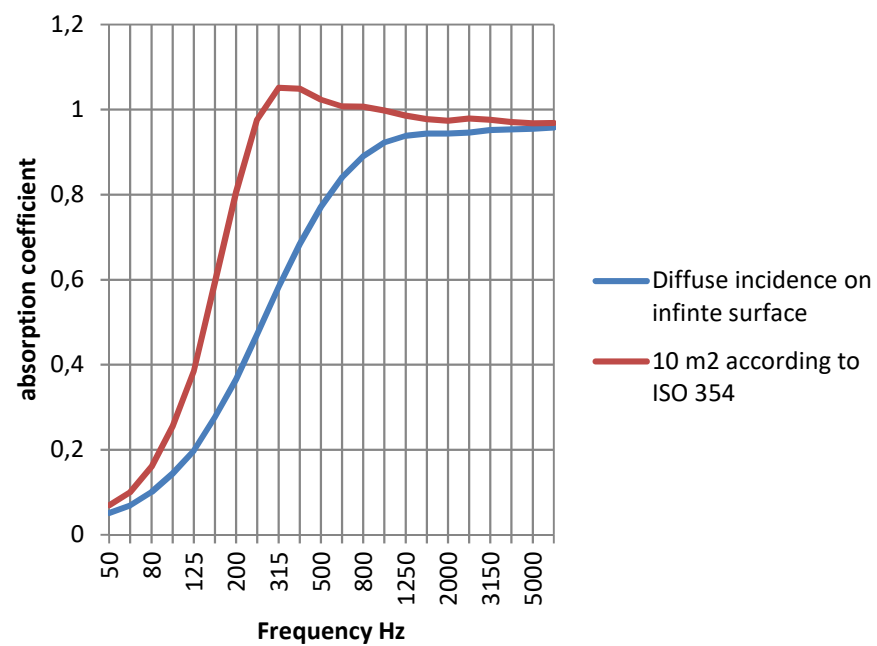
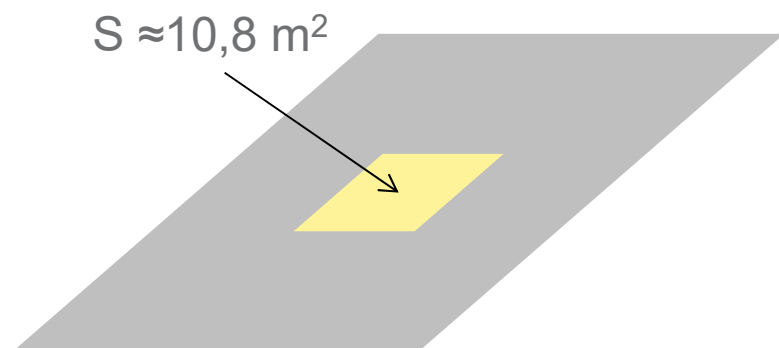
$$\alpha = \frac{0,16V}{S} \left(\frac{1}{T_{with}} - \frac{1}{T_{without}} \right)$$

Sound absorption coefficient



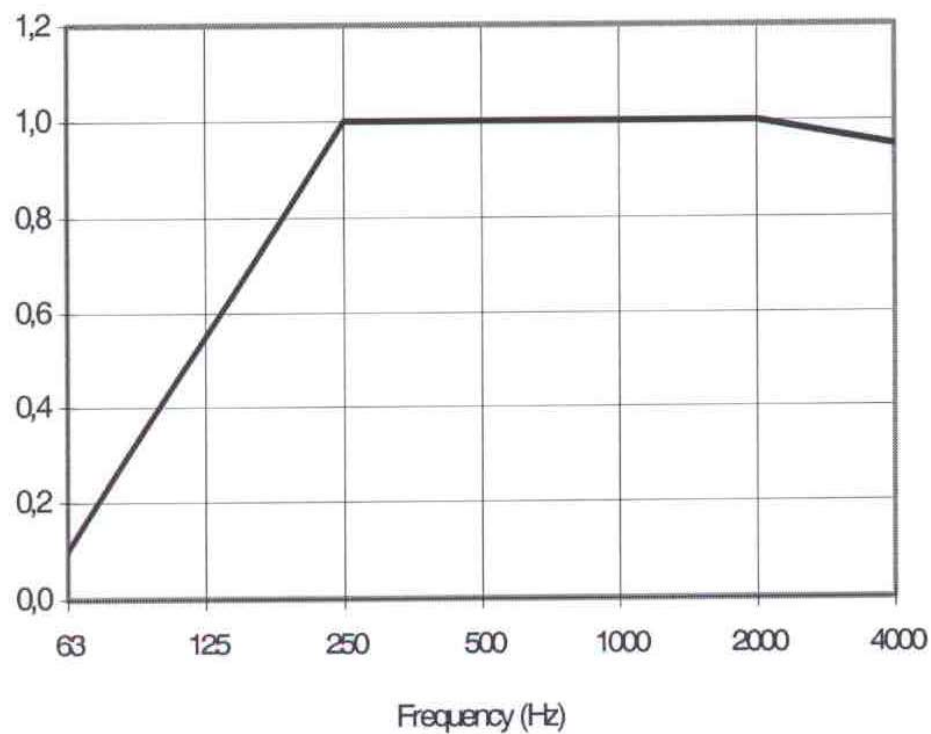
The area effect

- Sabine coefficient vs. random incidence



Practical sound absorption coefficient according to ISO 11654

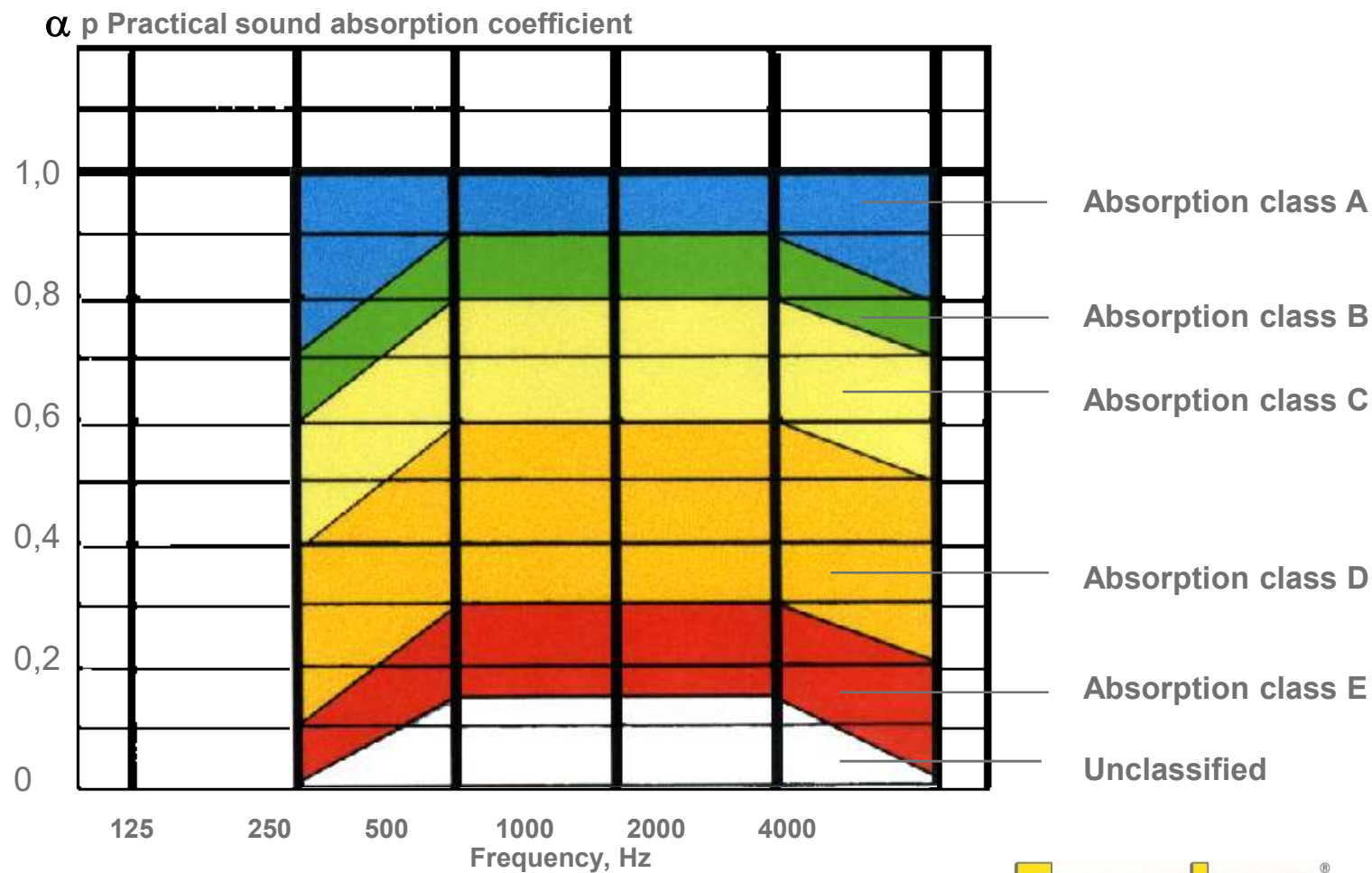
Practical sound absorption coefficient



Frequency (Hz)	α_p
63	0,10
125	0,55
250	1,00
500	1,00
1000	1,00
2000	1,00
4000	0,95

Classification of sound absorbers

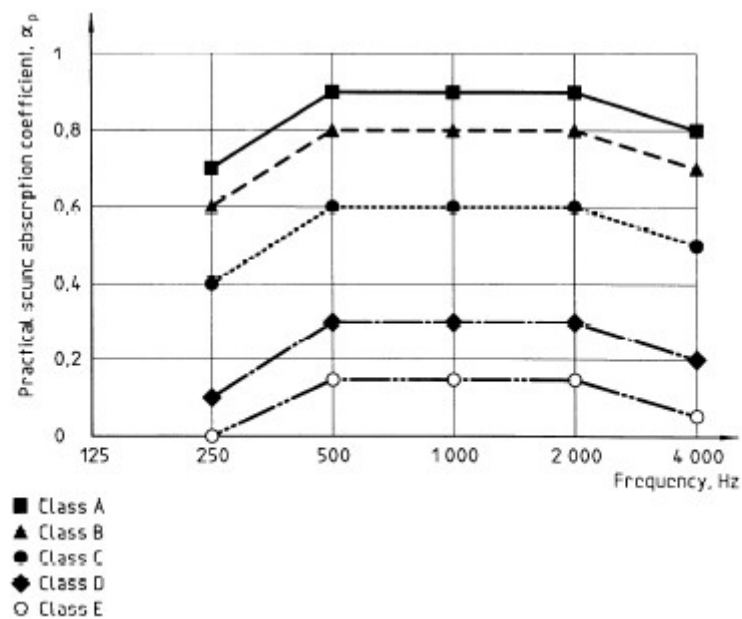
EN-ISO 11654



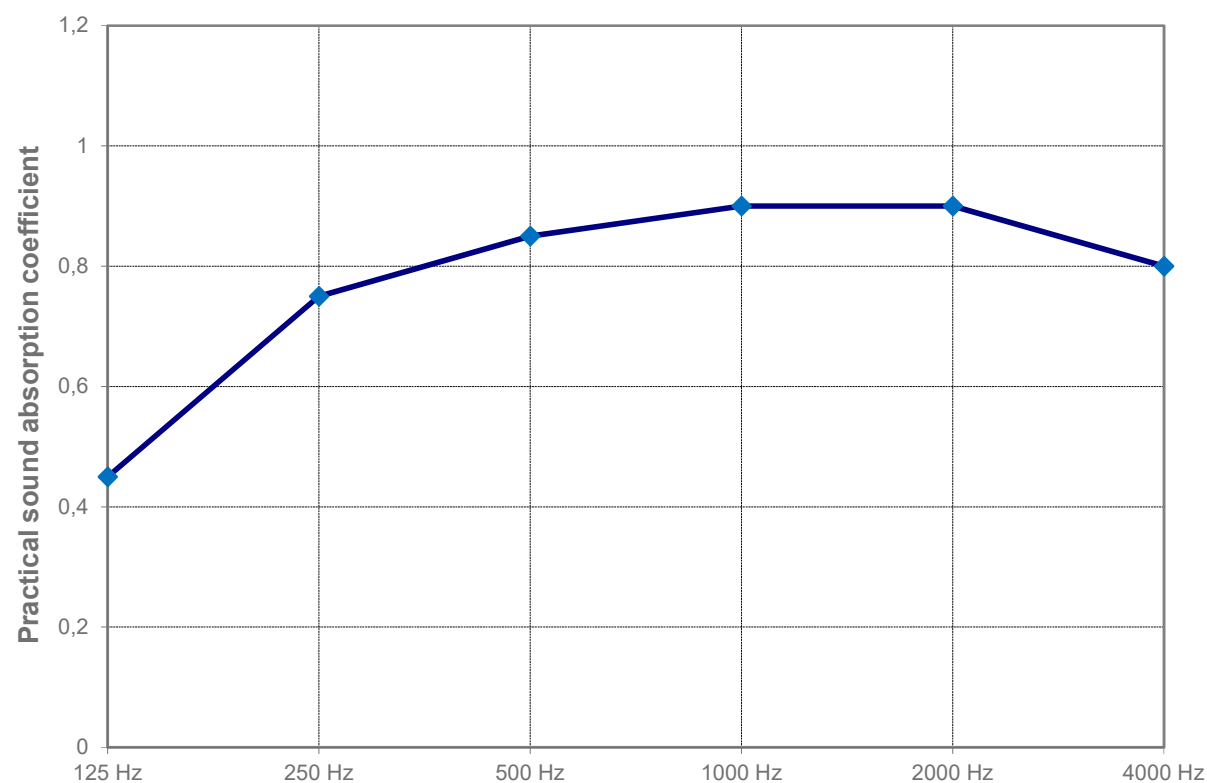
Weighted sound absorption coefficient according to ISO 11654

Table B.1 — Sound absorption classes

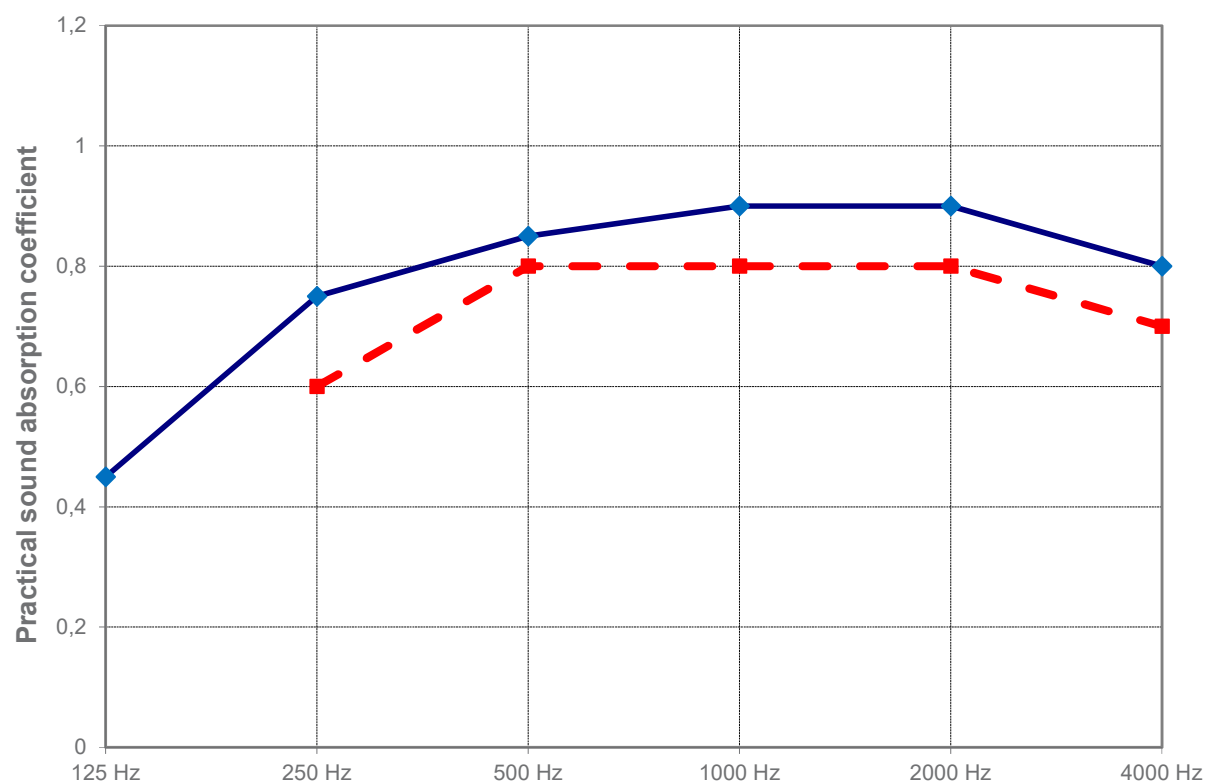
Sound absorption class	α_W
A	0,90; 0,95; 1,00
B	0,80; 0,85
C	0,60; 0,65; 0,70; 0,75
D	0,30; 0,35; 0,40; 0,45; 0,50; 0,55
E	0,25; 0,20; 0,15
Not classified	0,10; 0,05; 0,00



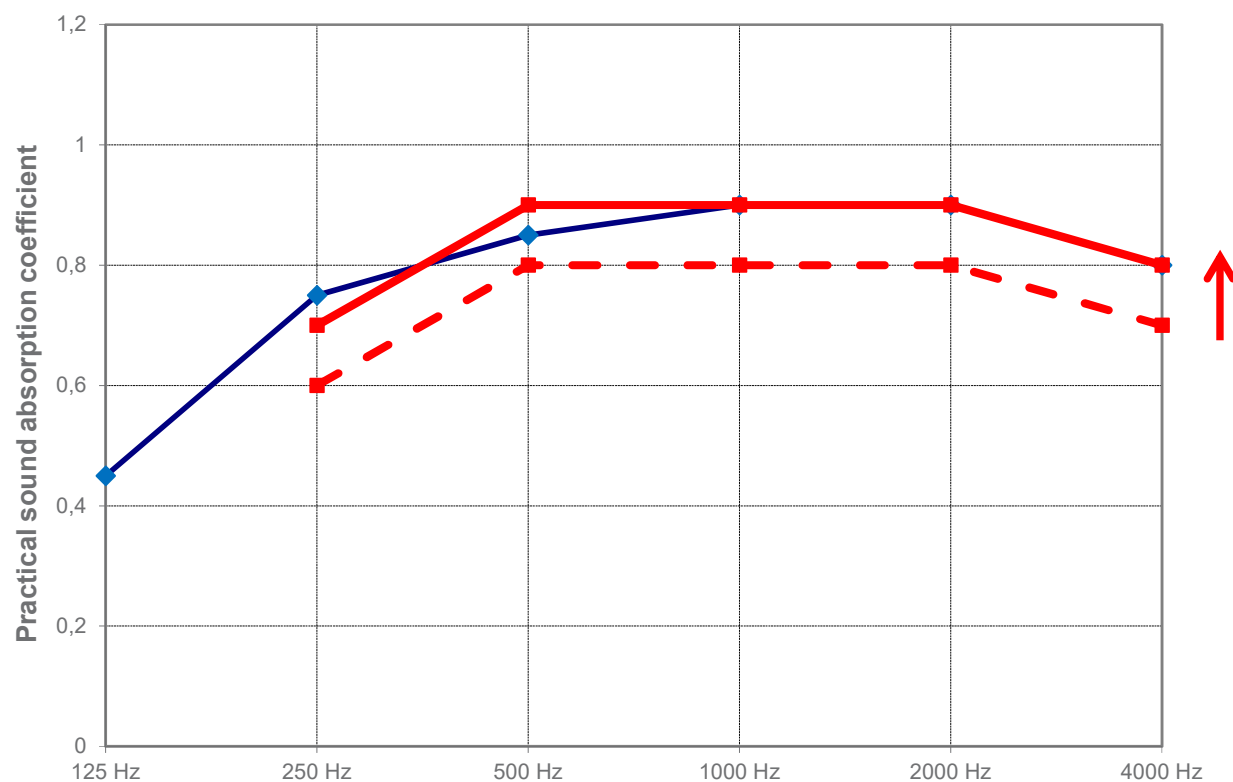
Practical absorption coefficient and classification



Practical absorption coefficient and classification



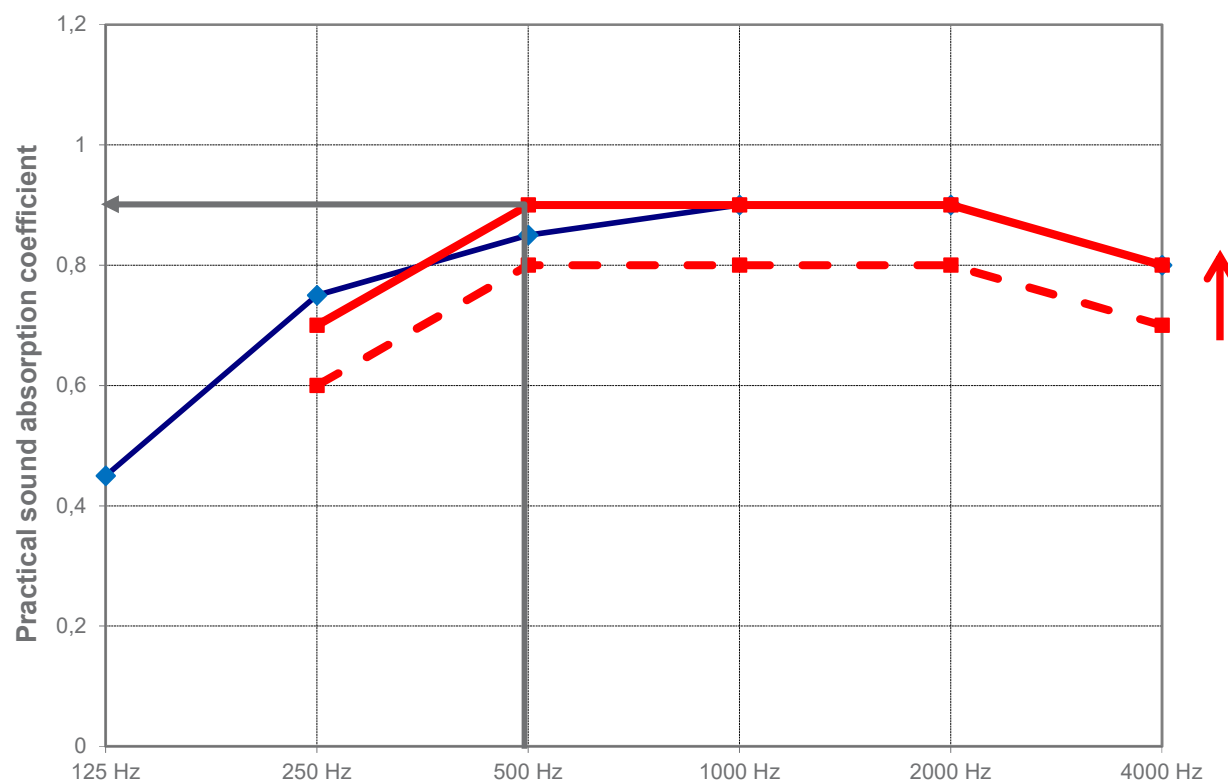
Practical absorption coefficient and classification



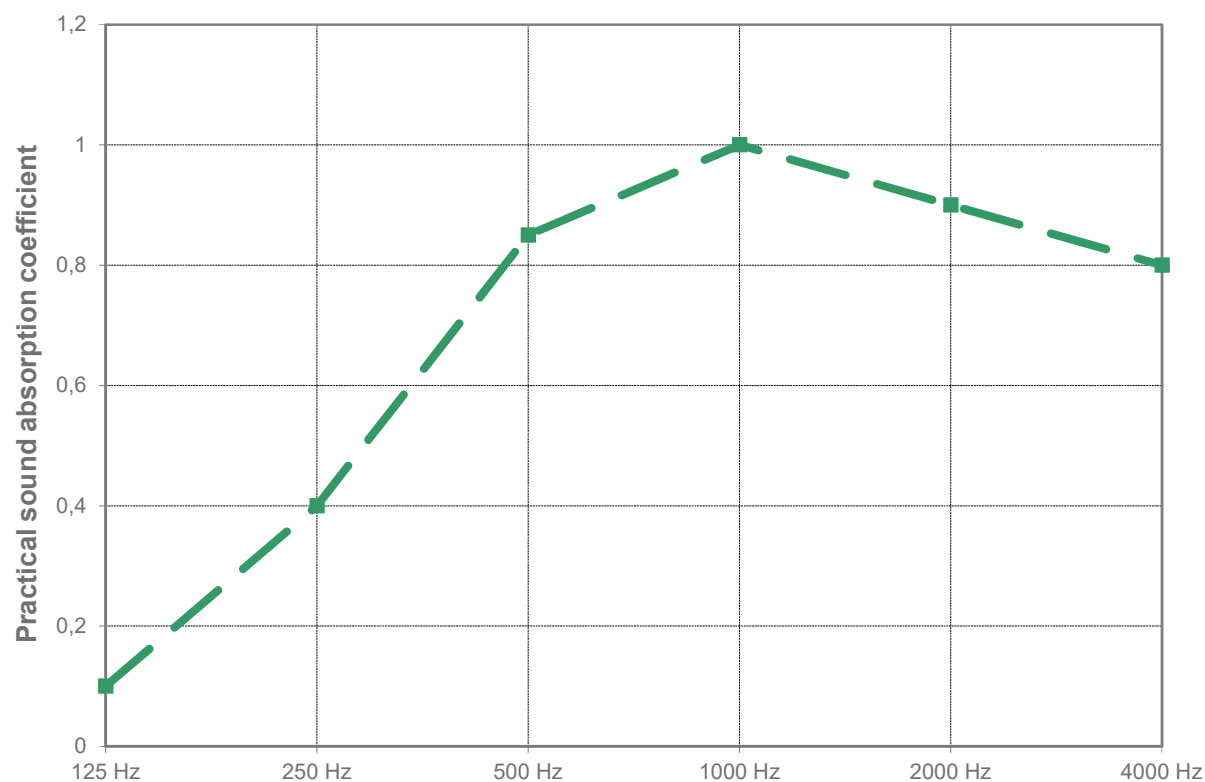
Practical absorption coefficient and classification

Weighted
absorption
coefficient, α_w

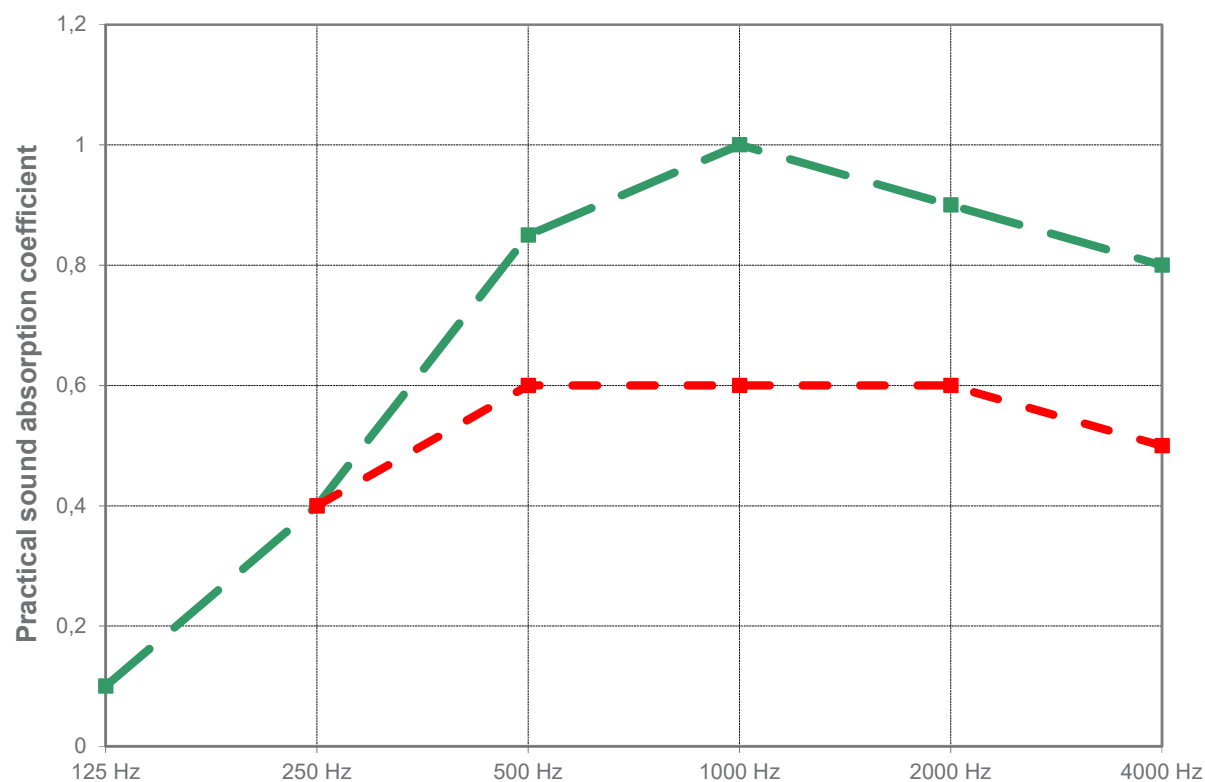
$\alpha_w = 0,9$, class A



Practical absorption coefficient and classification



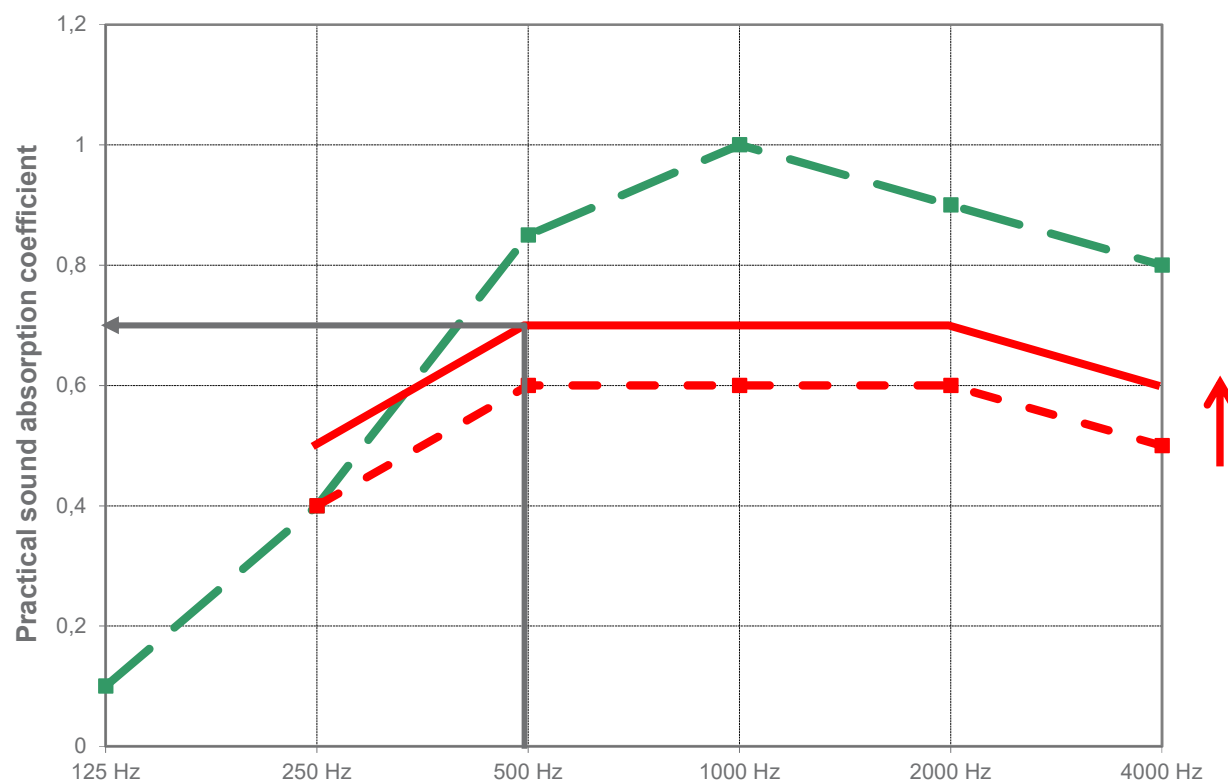
Practical absorption coefficient and classification



Practical absorption coefficient and classification

Weighted
absorption
coefficient, α_w

$\alpha_w = 0,7$, class C



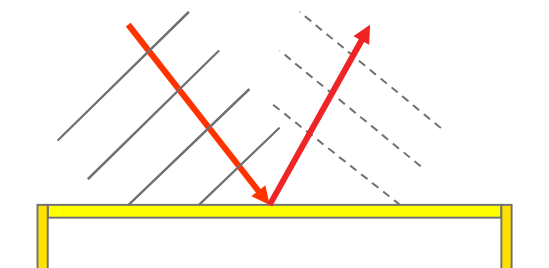
Use of practical absorption coefficients

- Comparison of products
- Sabine calculations
- Simulation software (Odeon, Catt, etc.)



FHU - Acoustic specification

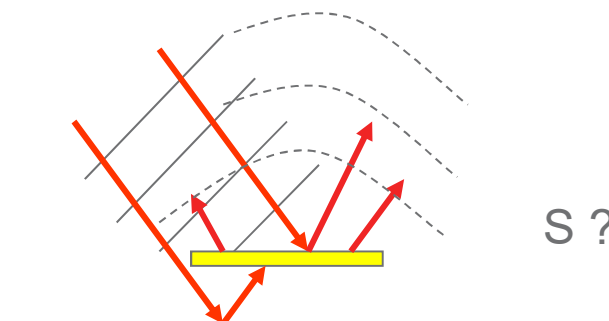
Absorbent ceiling



Absorption factor (ISO 354)

$$\alpha = \frac{A}{S}$$

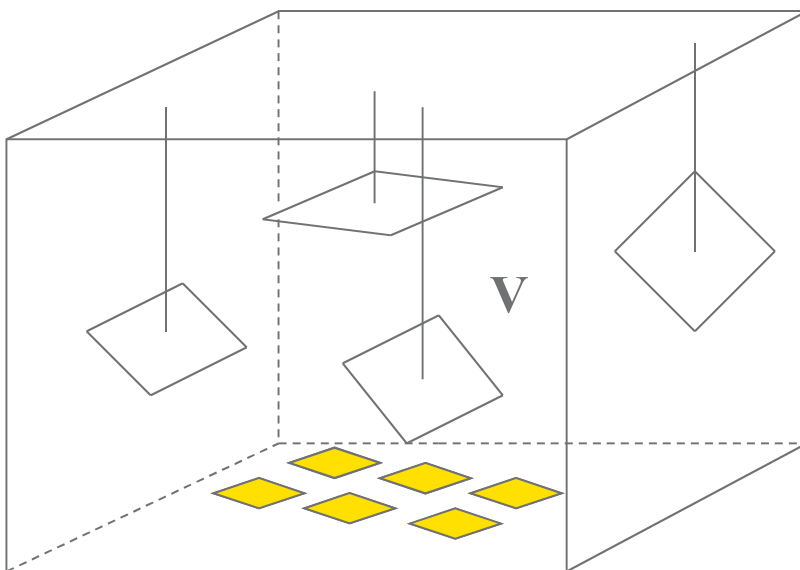
FHU – free hanging unit



Equivalent absorption area (ISO 354)

A (m²)

Free hanging units “Solo”

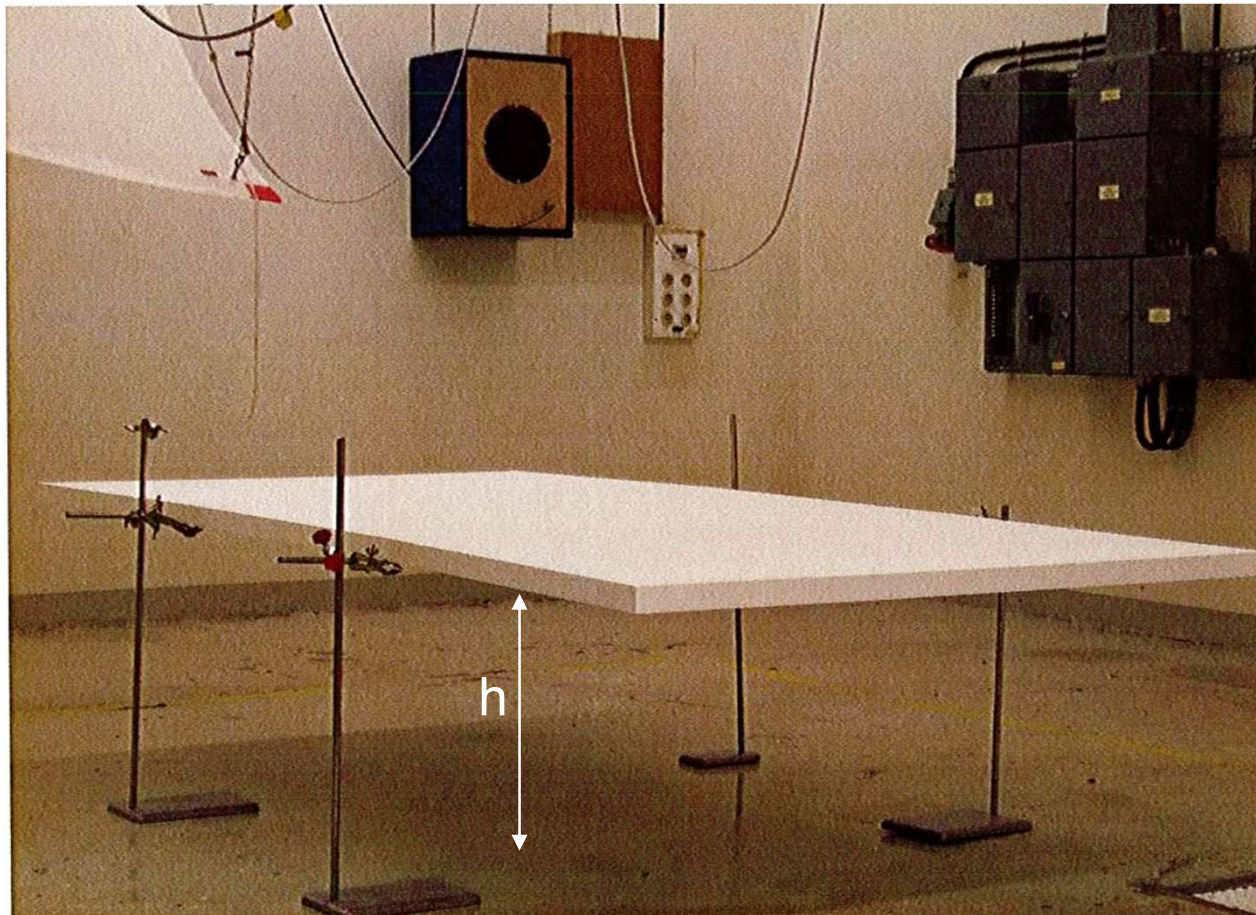


Reverberation chamber

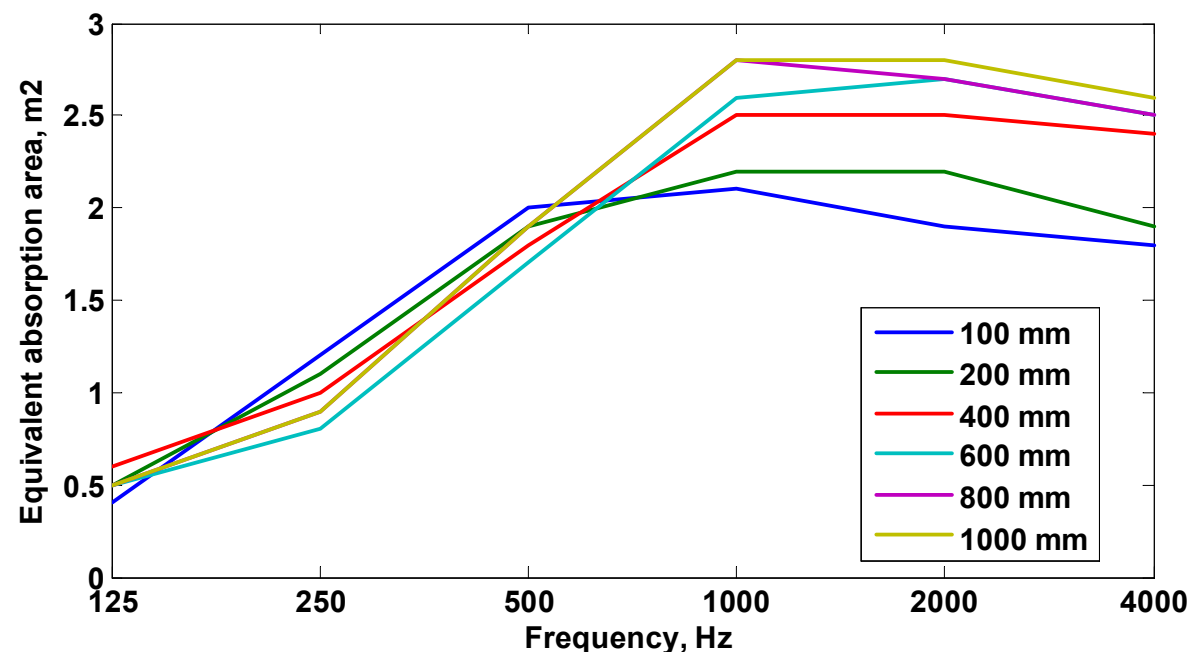
$$A = 0.16V \left(\frac{1}{T_{with}} - \frac{1}{T_{without}} \right)$$

$$A_{FHU} = A/6$$

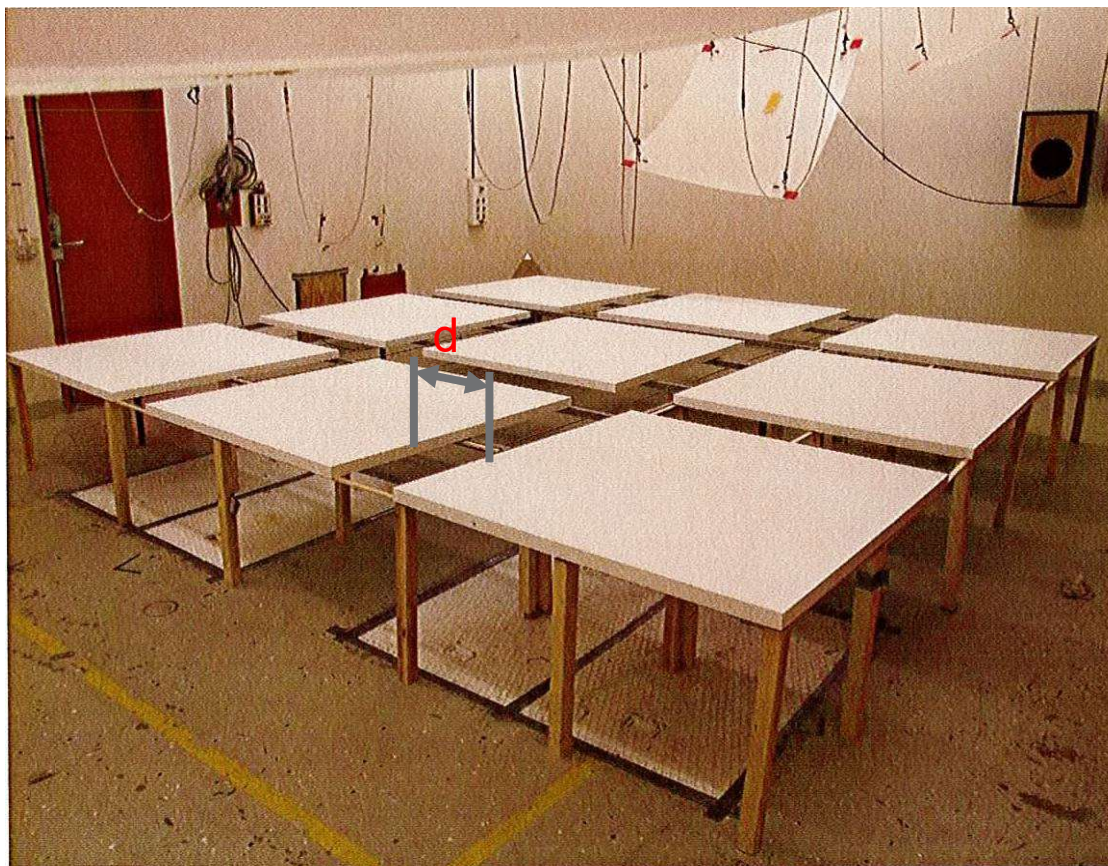
Master Solo S (2400x1200)



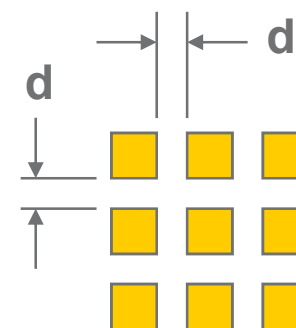
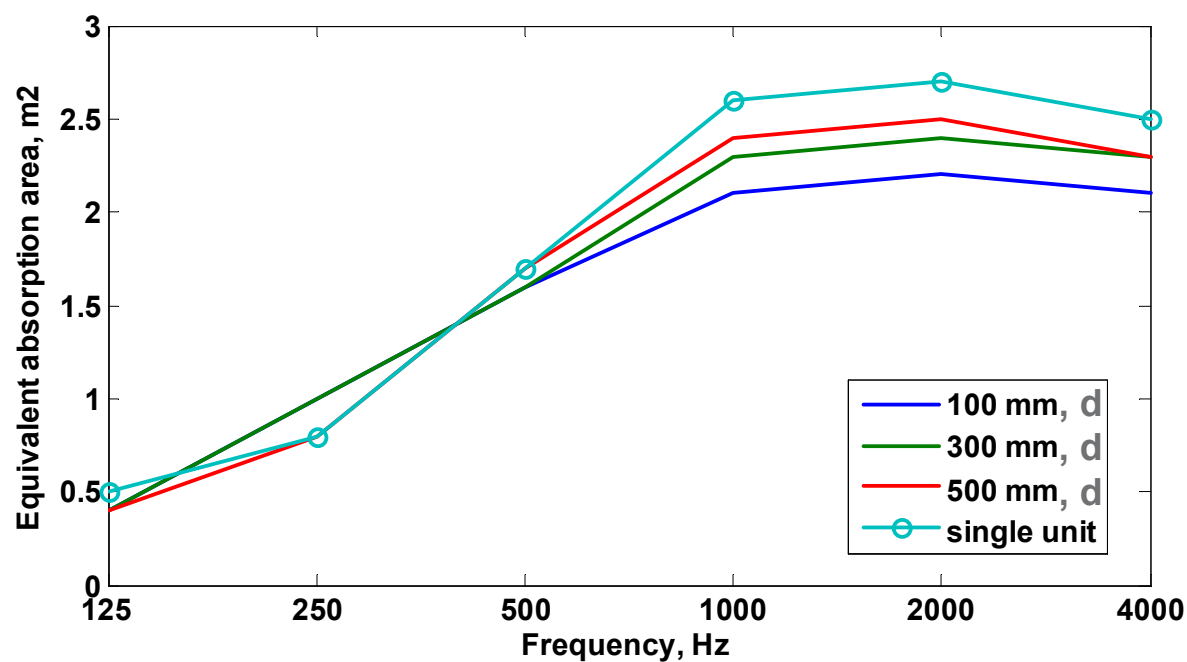
Master Solo S (1200x1200), single unit for various o.d.s.



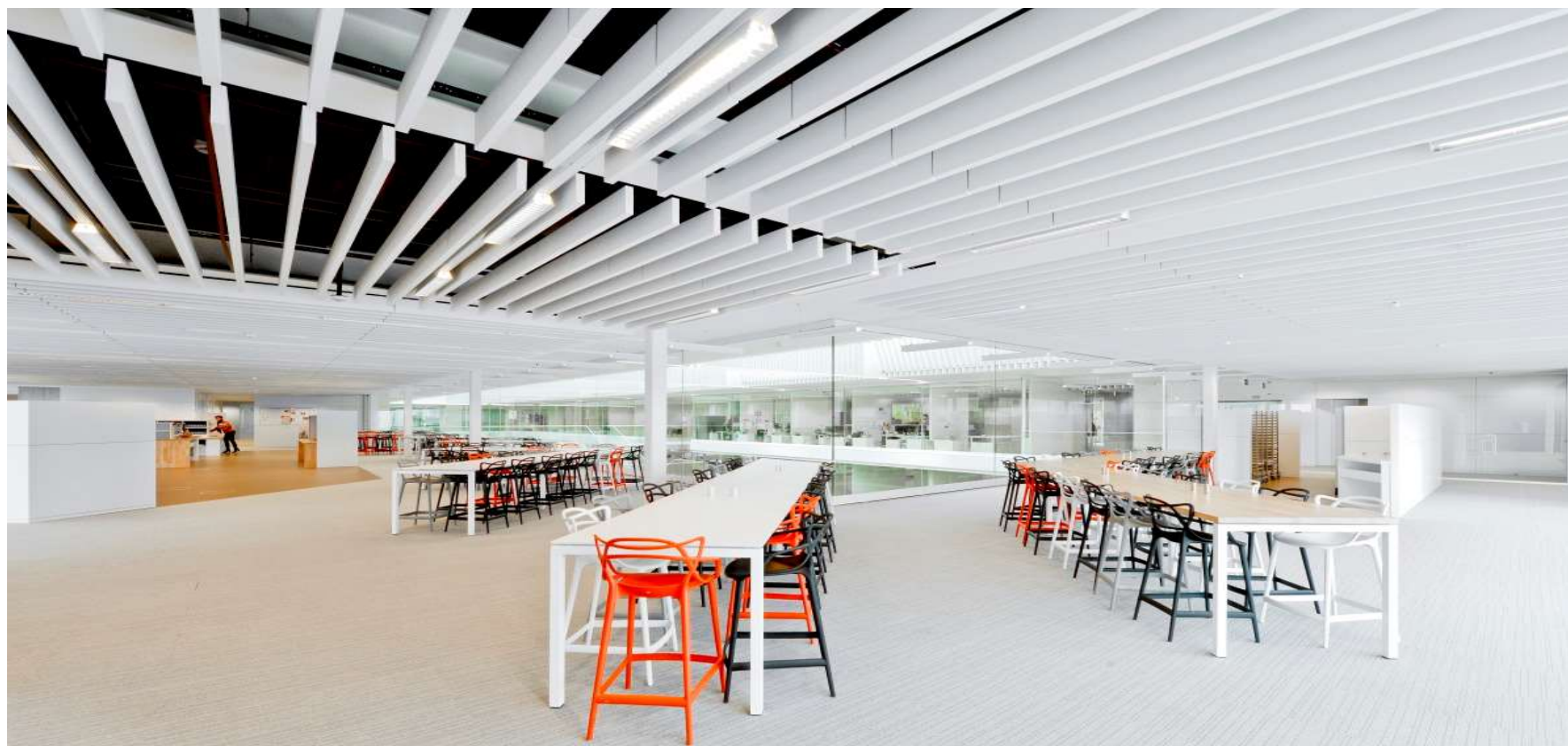
Array of Master Solo S (1200x1200)



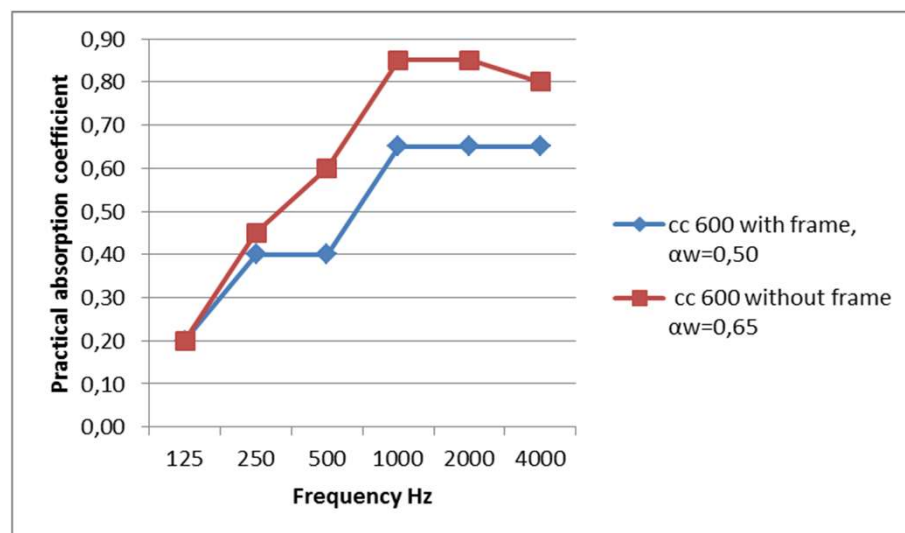
Master Solo S, array with 9 panels 1200 x 1200, 600 mm o.d.s., for various distances d



Solo Baffle: Laboratory measurements (see knowledge guide Solo™ Baffle - Acoustics)

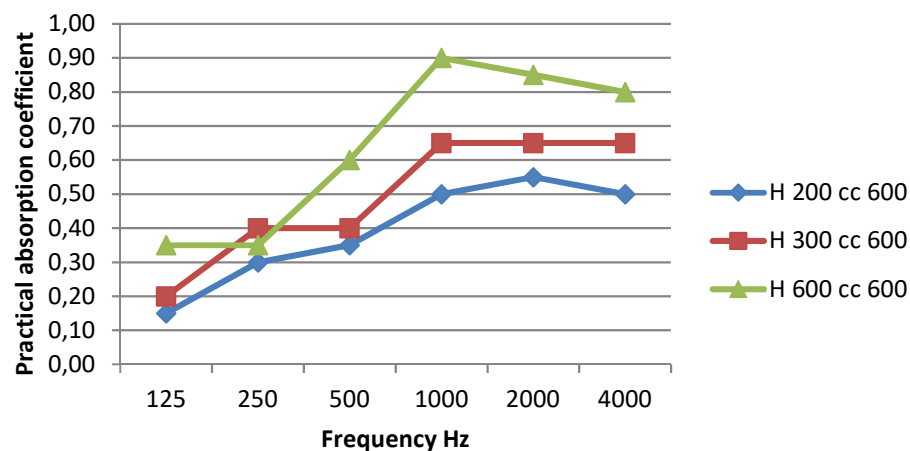


Laboratory measurements of baffles



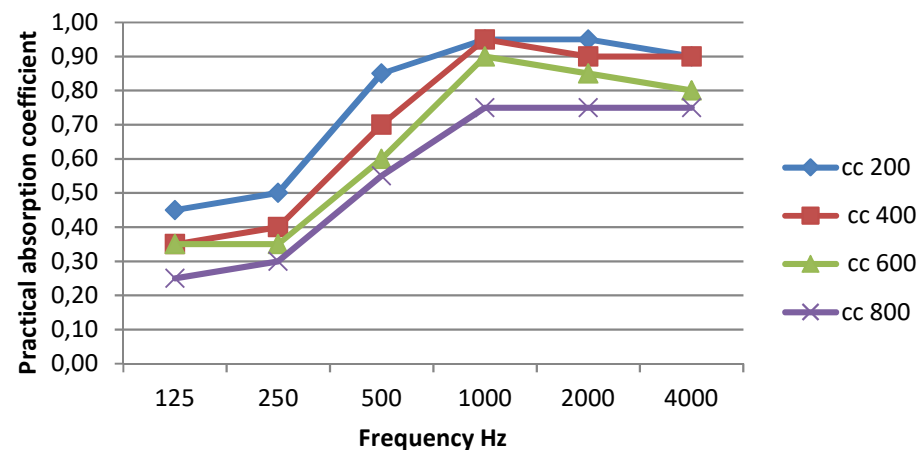
The effect of baffle height and the distance between baffles

The effect of baffle height



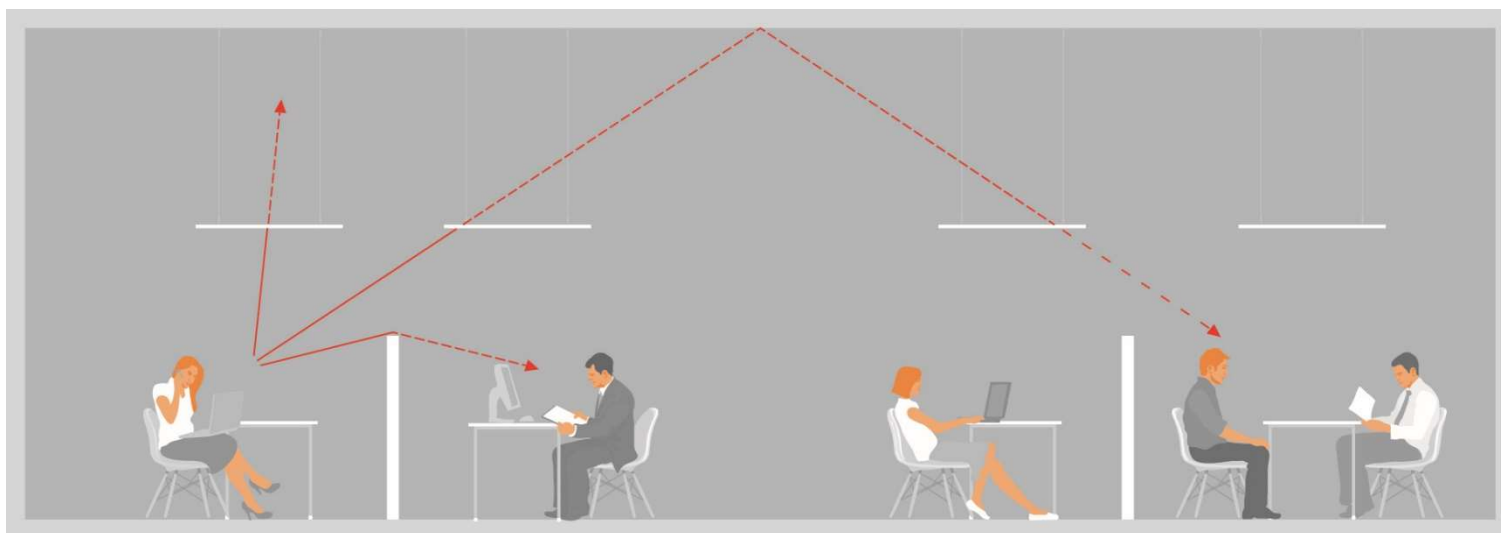
Practical absorption coefficient for Solo Baffle with heights 200, 300 and 600 mm at cc distance 600 mm

The effect of cc distance



Practical absorption coefficient for Solo Baffle 1200 x 600 x 40 at cc distances 200, 400, 600 and 800 mm

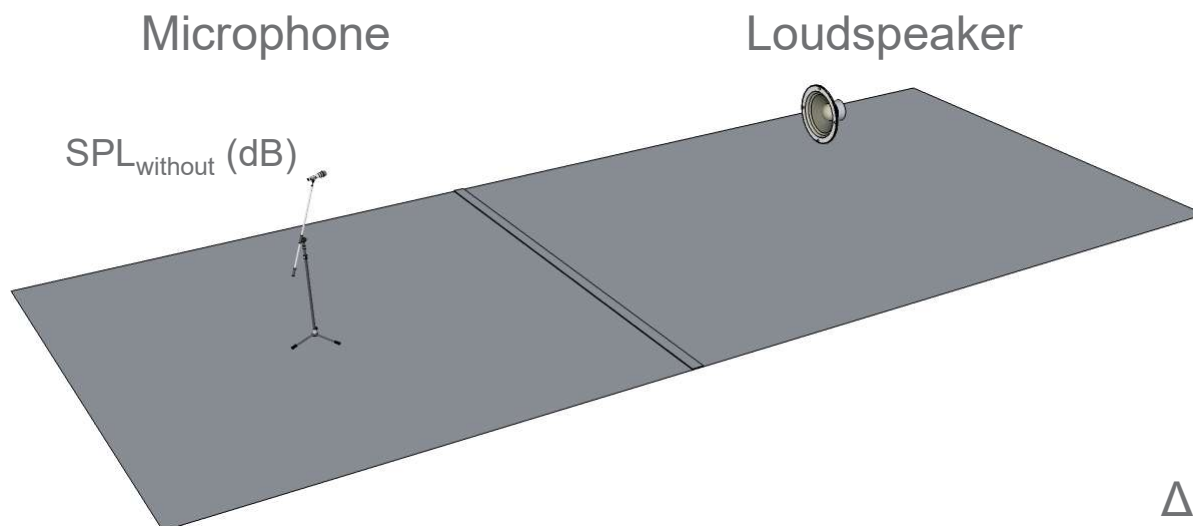
Speech privacy between workplaces in an open plan office



Screens in an open-plan office

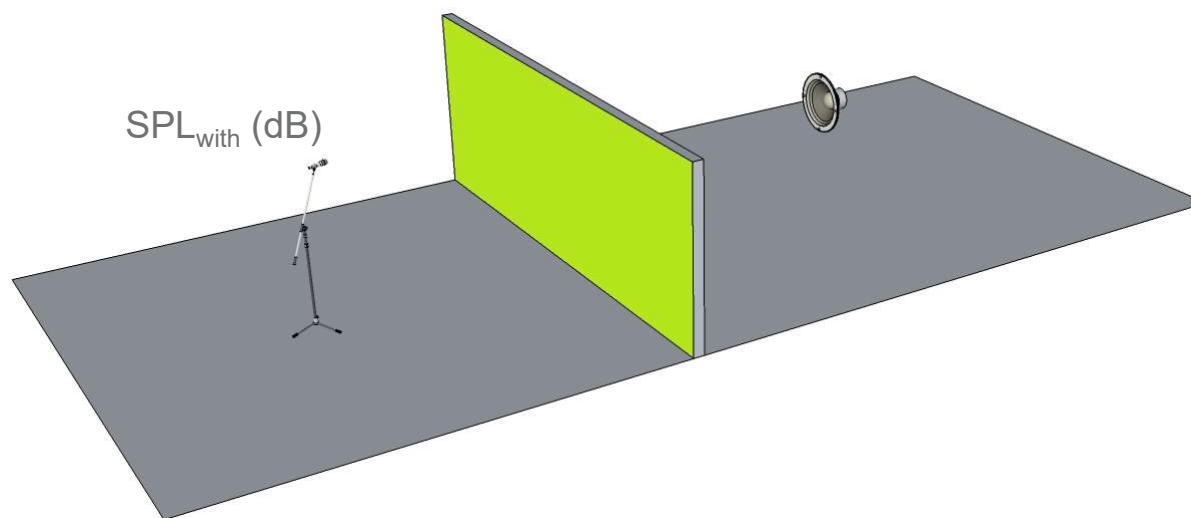


Screen attenuation (ISO 10053:1991)



Screen attenuation ΔL

$$\Delta L = SPL_{\text{without}} - SPL_{\text{with}} \text{ (dB)}$$



Screen attenuation and its dependence of screen height. Calculated values for outdoor conditions

